

- Techniques de mesure  
(taux de vide)
- Modèles de taux de vide.  
Calcul pratique.

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## RAPPELS des OPERATEURS de MOYENNE.

- Fonction indicatrice de phase

$$X_k(r, t) = 1 \text{ si } M(r) \in \text{phase } k$$

- Moyennes spatiales

phasique :  $\langle f \rangle_n = \frac{1}{\mathcal{D}_{kn}} \int_{\mathcal{D}_{kn}} f d\mathcal{D}_{kn}$

globale :  $\langle f \rangle_n = \frac{1}{\mathcal{D}} \int_{\mathcal{D}} f d\mathcal{D} \quad (\mathcal{D} = \cup_k \mathcal{D}_k)$

- Moyennes temporelles

conditionnelle :  $\overline{f}^x = \frac{1}{T_k} \int_{[T_k]} f dt$

globale :  $\overline{f} = \frac{1}{T} \int_{[T]} f dt$

$[T]_k$  ensemble des instants où  $X_k = 1 \cap [T]$

$[T]$  intervalle d'intégration  $[t - \frac{T}{2}, t + \frac{T}{2}]$

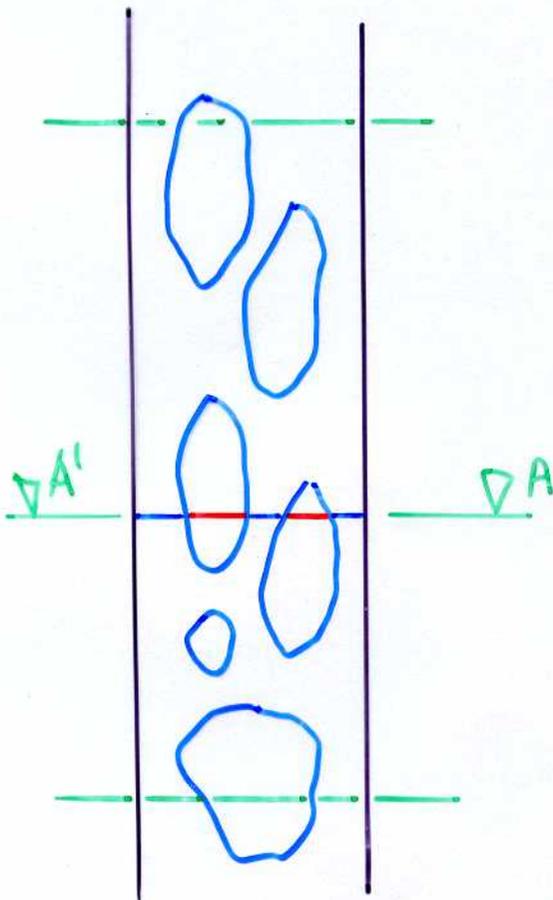
**IDENTITÉ :**  $R_{kn} \langle f \rangle_n = \langle \alpha_k \overline{f}^x \rangle_n$

# LES DEFINITIONS du TAUX de VIDE ( $\alpha$ )

- Taux de vide local  $\alpha_G$

temps de présence moyen :  $\alpha_G \triangleq \frac{T_G}{T_G + T_L} = \frac{T_G}{T}$

- Fractions Spatiales de gaz/vapeur/vide instantanées



- fraction linéique instantanée

$$R_{G1} \triangleq \frac{L_G}{L_G + L_L} = \frac{L_G}{L}$$

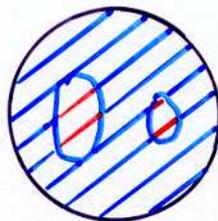
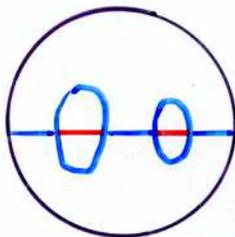
- fraction surfacique instant.

$$R_{G2} \triangleq \frac{A_G}{A_G + A_L} = \frac{A_G}{A}$$

- fraction volumique instant.

$$R_{G3} \triangleq \frac{V_G}{V_G + V_L} = \frac{V_G}{V}$$

Coupe  
A-A'



## RELATIONS FONDAMENTALES IDENTITÉS

- Rappel Commutativité des opérateurs

$$\overline{R_{kn} \langle f \rangle_n} = \langle \alpha_k^x f \rangle_n \quad (\text{avec } f \equiv 1)$$

- Taux de présence (vide) moyen

+ sur une ligne

$$\overline{R_{G1}} \triangleq \frac{1}{T} \int_{[T]} R_{G1} dt \equiv \frac{1}{L} \int_L \alpha_G dL$$

+ sur une surface

$$\overline{R_{G2}} \triangleq \frac{1}{T} \int_{[T]} R_{G2} dt \equiv \frac{1}{A} \int_A \alpha_G dA$$

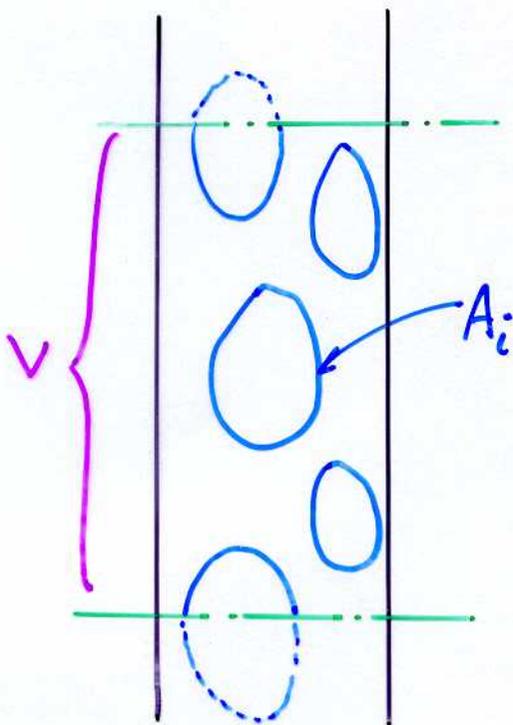
+ dans un volume

$$\overline{R_{G3}} \triangleq \frac{1}{T} \int_{[T]} R_{G3} dt \equiv \frac{1}{V} \int_V \alpha_G dV$$

- Les définitions sont précises.

## AUTRES DEFINITIONS

- Aire interfaciale volumique (instantanée)



$$\overline{\Gamma}_3 \triangleq \frac{A_i}{V}$$

- Aire interfaciale locale

$$\gamma = \sum_{\substack{\text{discont.} \\ \in [T]}} \frac{1}{|v_i \cdot n_k|}$$

- Identité :

$$\overline{\Gamma}_3 \equiv \langle \gamma \rangle_3$$

## TECHNIQUES de MESURE du TAUX de VIDE.

- Taux de vide local

Sondes électriques }  $\alpha$  et  $\gamma$   
Sondes optiques }

- Taux de présence sur une ligne

Atténuation des Rayons X ou  $\gamma$ .

- Taux de présence surfacique

Rayons X ou  $\gamma$  : one-shot technique

Densitomètre multi faisceaux

Diffusion de neutrons

Densitomètre à impédance (ep. de film)

- Taux de vide moyen (volumique)

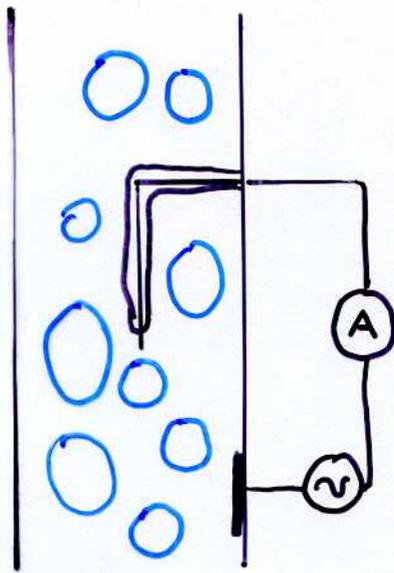
Vannes à fermeture rapide

Méthodes ultrasonores ( $\alpha$ ,  $\beta$ ,  $d_{sm}$ )

# TAUX de VIDE LOCAL

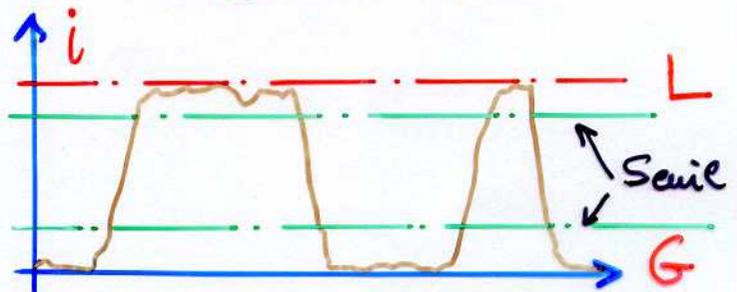
Mesure de la FIP ( $X_R(r, t)$ )

- Sondes résistives (résistivité)

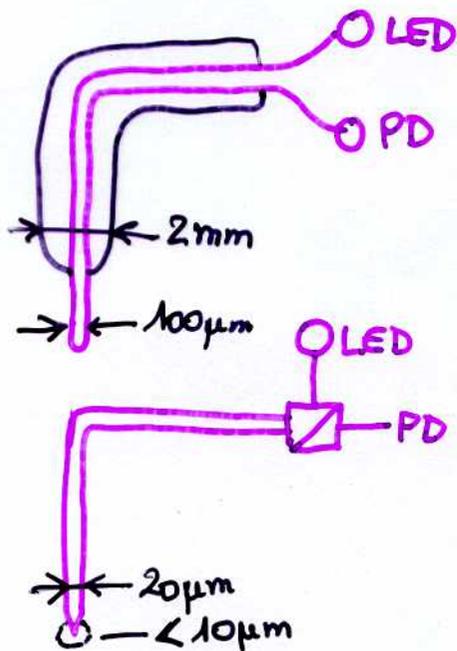


Milieu continu conducteur  
Phase dispersée volatile

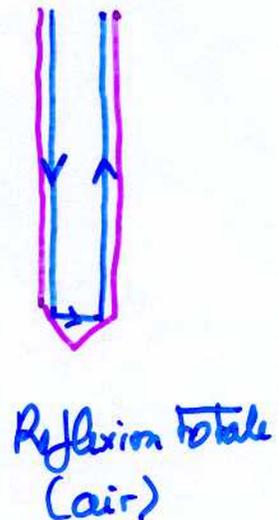
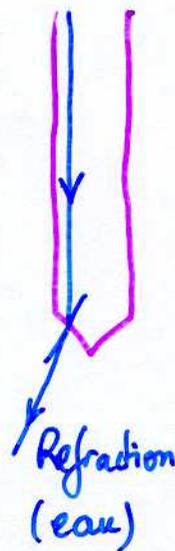
Seuillage :  $\alpha$  (seuil)



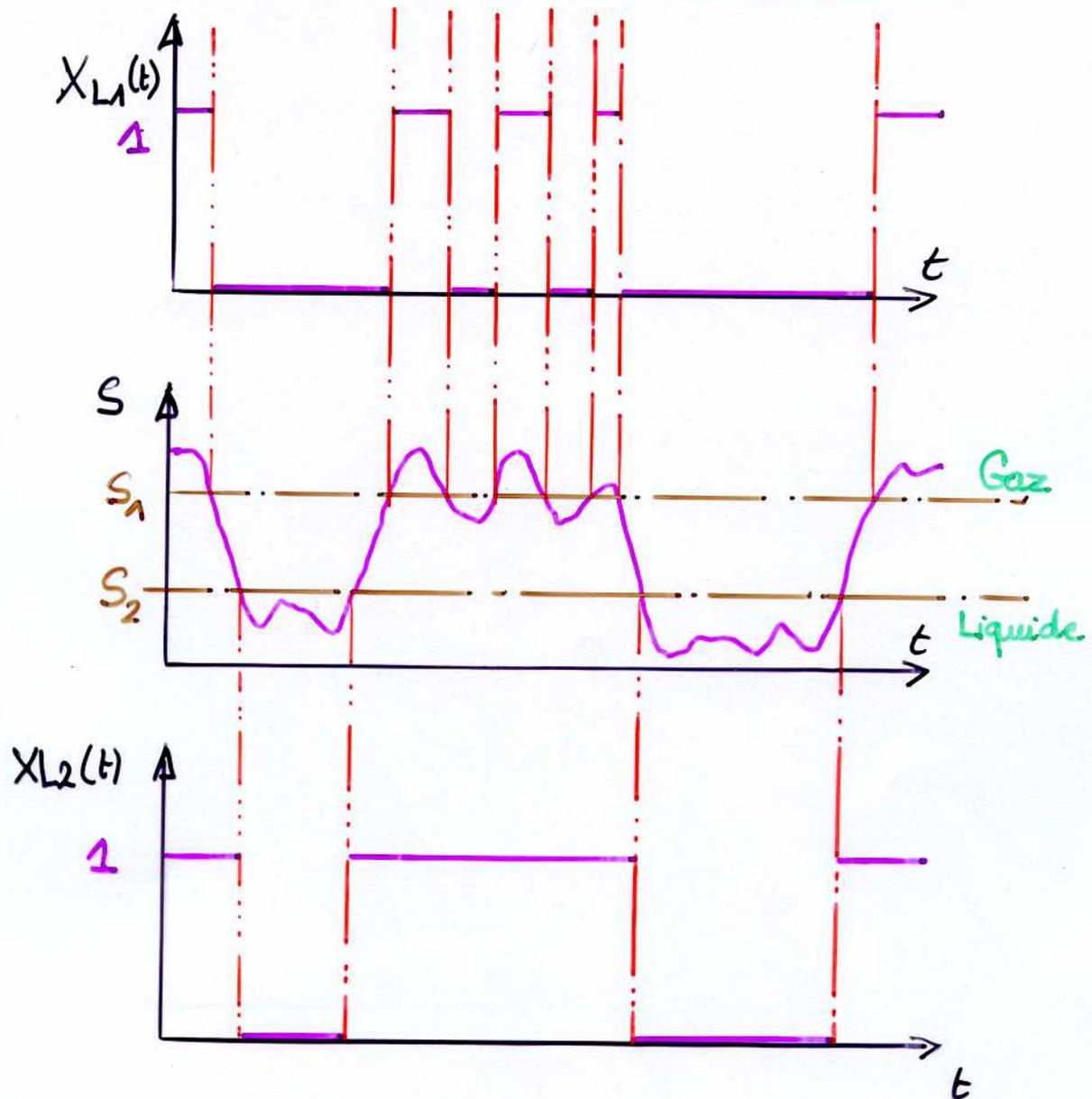
- Sondes optiques (indice optique) eau  
freon  
 $T < 110^\circ\text{C}$



Pointe de la fibre



# DETERMINATION des SEUILS



- Détermination du seuil  $s$ .  $\alpha_1 < \alpha_2$  !

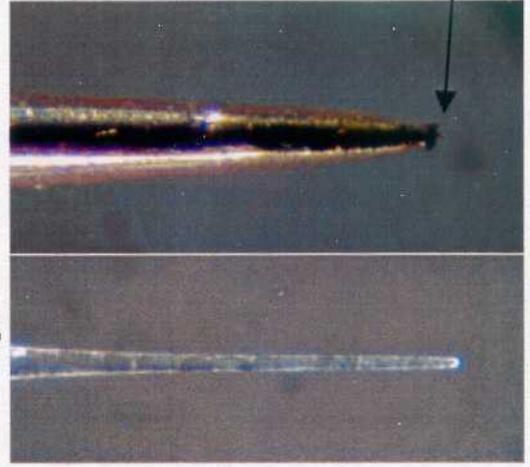
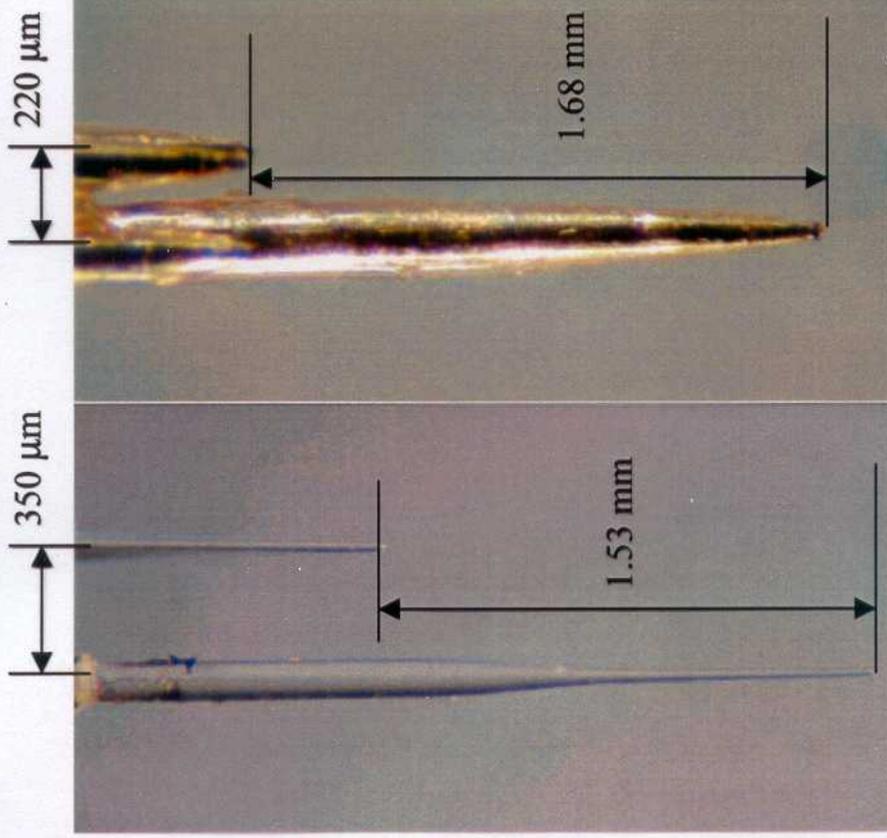
Méthode de référence ex  $\Delta P \rightarrow \overline{R_G}$

$$\alpha_G(s) = \frac{T_G(s)}{T} ; \text{profil ds une conduite}$$

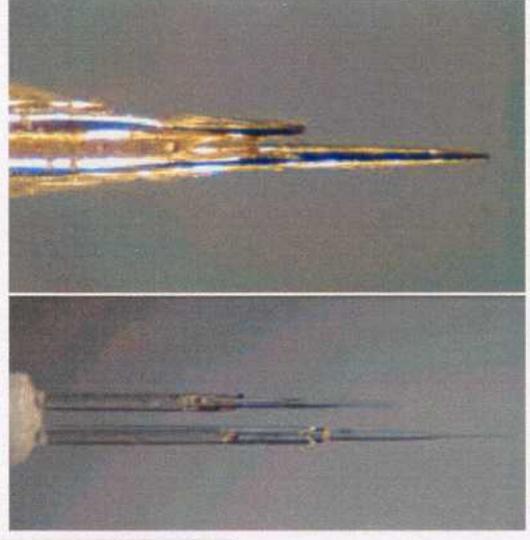
$$\text{Ajustement } s / \langle \alpha_G(s) \rangle = \overline{R_G}$$

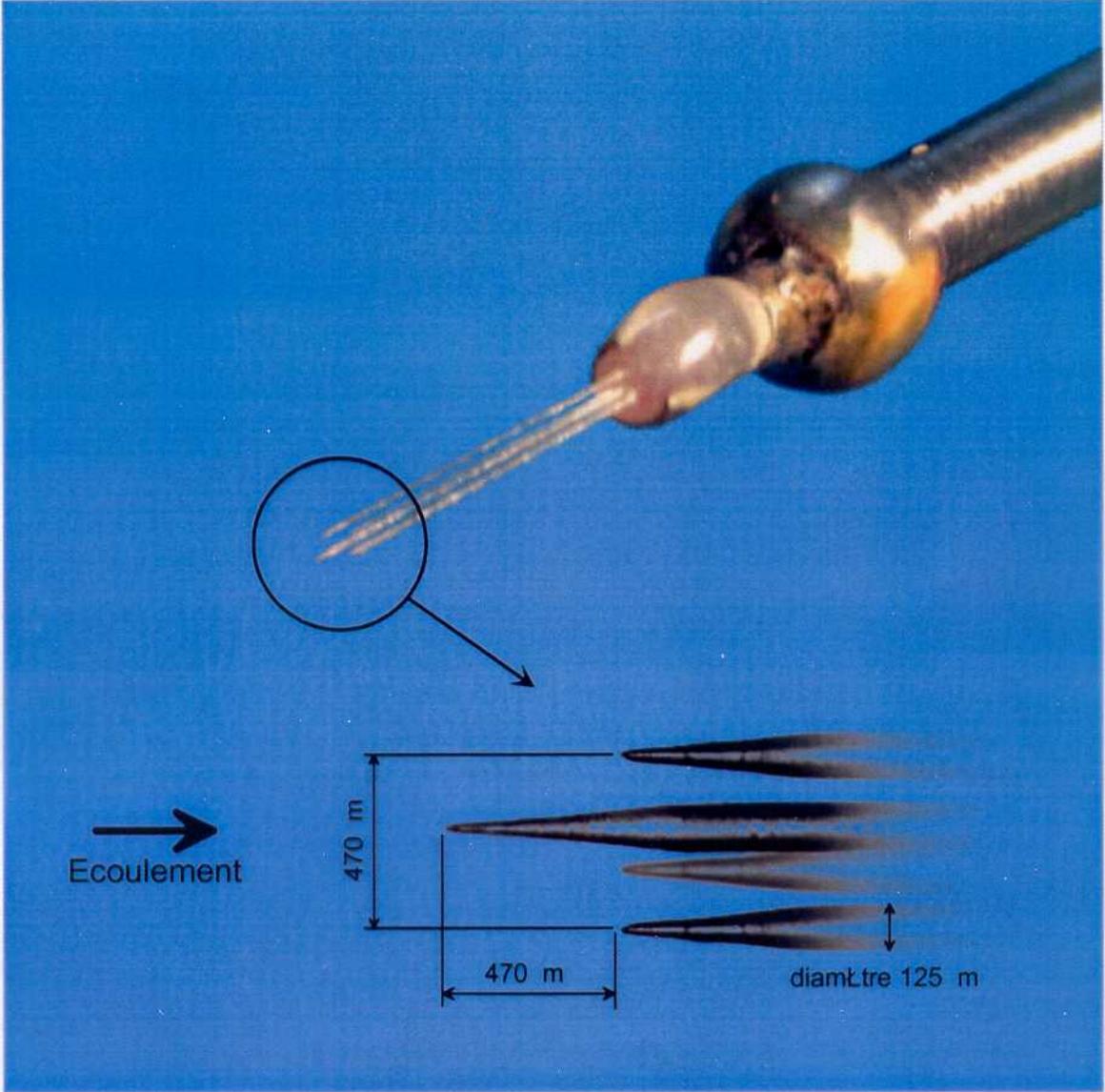
# Optical vs Electrical four-sensor probe

$\phi_{\text{cond}} \approx 200 \mu\text{m}$   
 $\phi_{\text{opt}} \approx 50 \mu\text{m}$



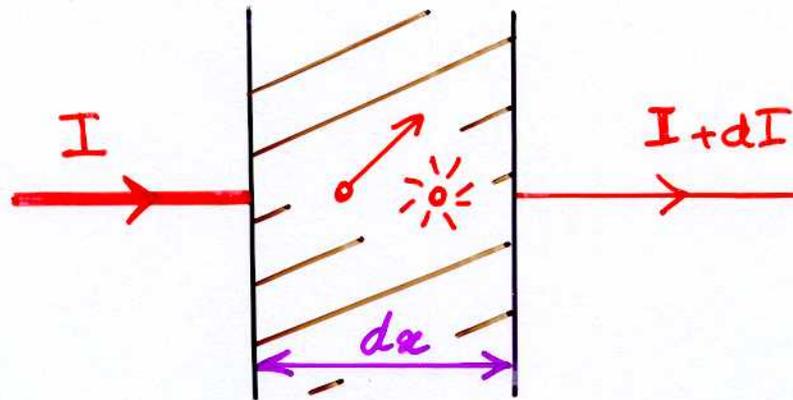
Overall view





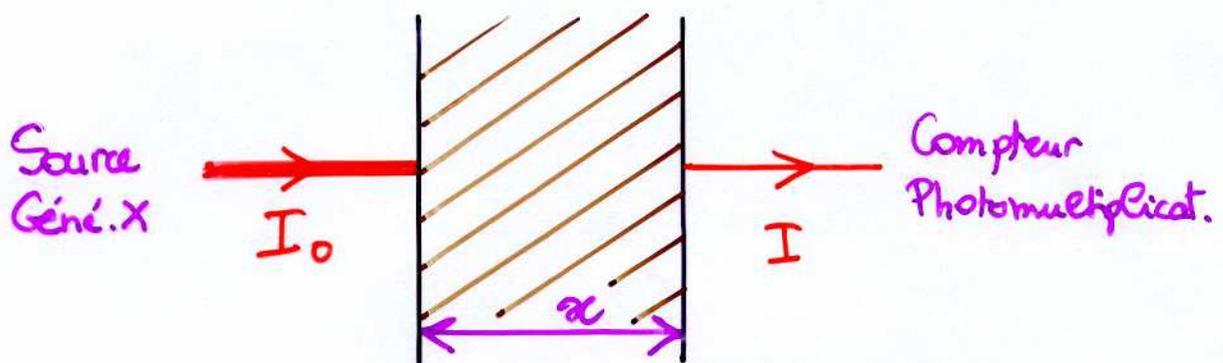
# ATTENUATION RX en $\gamma$

## Méthodes Photoniques



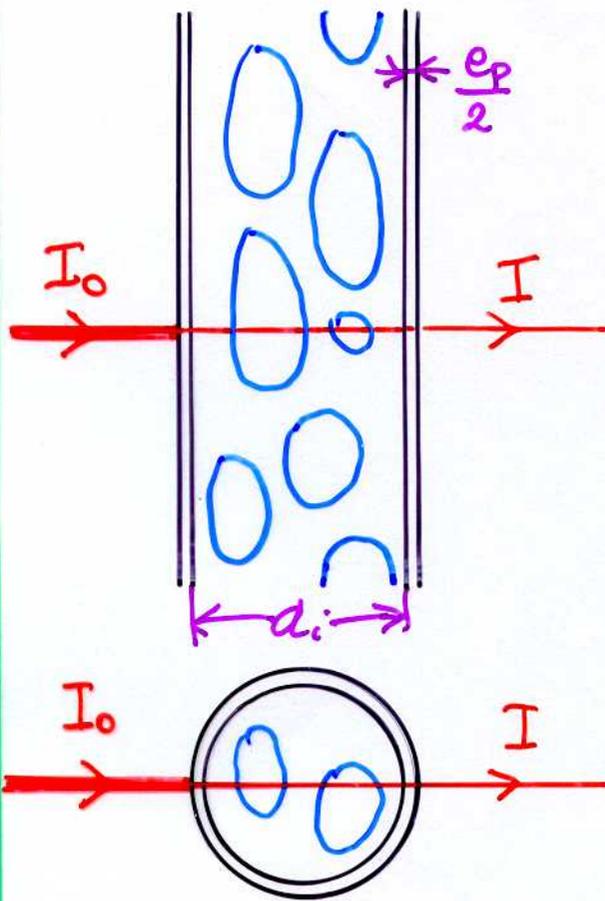
loi de Beer-Lambert :  
faisceau collimaté, monoenergie (raie)

$$dI = -\mu I dx \quad [\mu] = L^{-1}$$



$$I = I_0 \exp(-\mu x) = I_0 \exp\left(-\left(\frac{\mu}{\rho}\right) \rho x\right)$$

# FRACTION LINÉAIRE de Phase



- Loi de Beer Lambert

$$I = I_0 \exp(-\mu_p e_p) \cdot \exp(-\mu_L (1-R_{G1}) d_i) \cdot \exp(-\mu_G R_{G1} d_i)$$

$$R_{G1}(z, t) \triangleq \frac{L_G}{L_G + L_L} = \frac{L_G}{d_i}$$

- Approximation Base Proportion

Monophasique Gaz:  $I_G = I_0 \exp(-\mu_p e_p)$

Monophasique Liquide:  $I_L = I_0 \exp(-\mu_p e_p) \exp(-\mu_L d_i)$

Écoulement diphasique:  $I = I_0 \exp(-\mu_p e_p) \exp(-\mu_L (1-R_{G1}) d_i)$

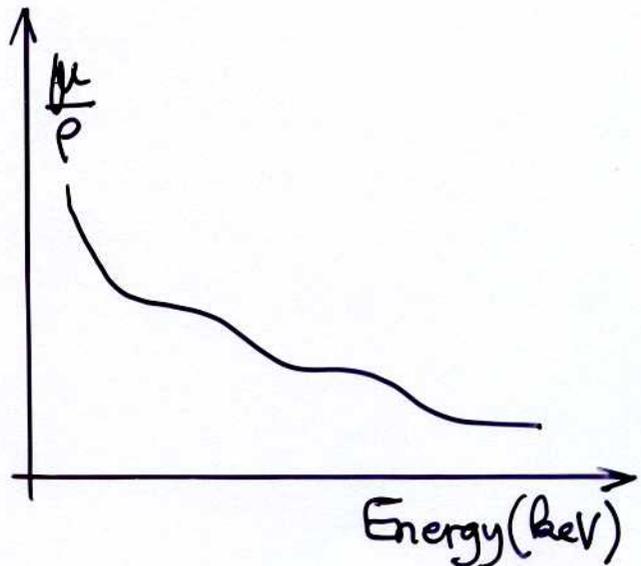
Élimination  $I_0, \mu_p e_p \Rightarrow R_{G1} = \frac{\log I/I_L}{\log I_G/I_L}$

# SOURCES D'ERREURS

- Contraste

$$\frac{I_G}{I_L} = \exp\left(\frac{\mu_L}{\rho_L}\right) \rho_L d_i$$

→ travailler à Base En



- Erreurs statistiques

$$I \propto N; \frac{\Delta N}{N} = \sqrt{\frac{1}{N}} \rightarrow \text{travailler à haute Energie}$$

- Fluctuations du taux de vide

$$\text{Mesure de } [I] = \int_0^z I dt \quad \underline{\underline{\text{et}}} \quad I(t) \propto \exp R_G(t)$$

$\exp f \neq \exp \bar{f}$  (non linéarité!)

ex: écoulement à poches  $\Delta R_G \sim 0,20$ , agité  $\Delta R_G \sim 0,05$

- Stabilité de la source

$I_0(t) \rightarrow$  méthode à faisceau de ref.  $I \rightarrow \frac{I}{I_0}$

- Durcissement du spectre (f. multi-énergie)

filtres, Spectrométrie, étalonnage direct

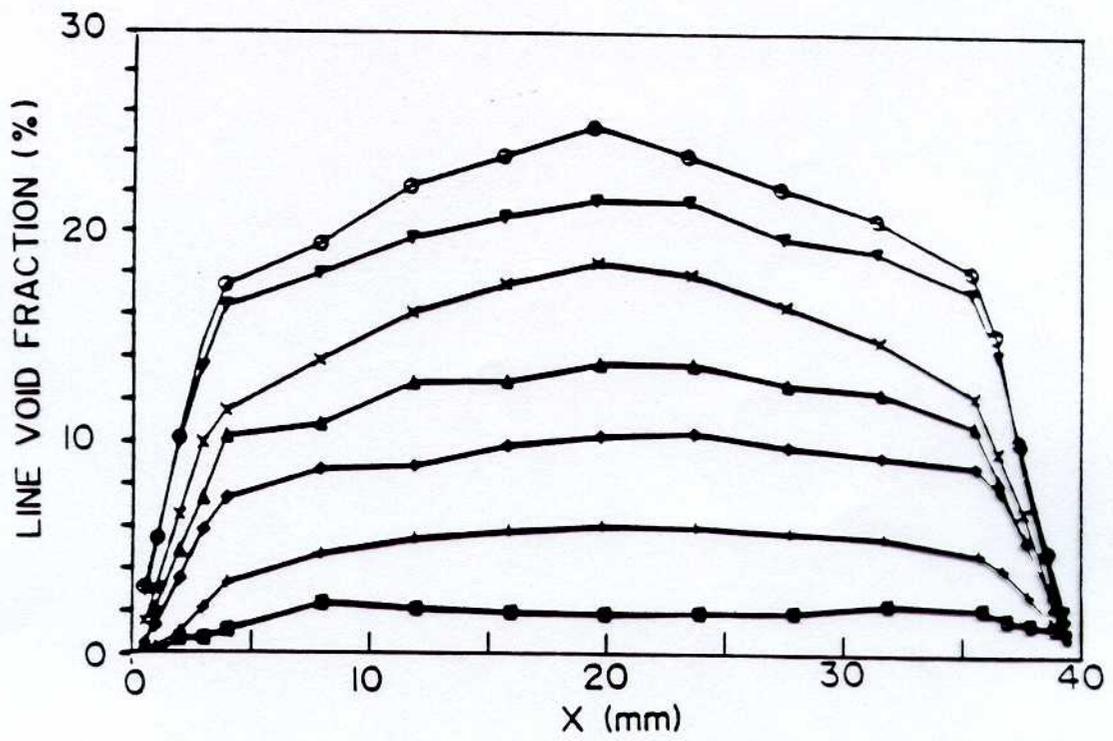


Figure 11 : Void fraction profiles for stagnant flow ( $v_1 = 0$ ,  $L = 40$  mm),  
( $\square \epsilon = 0.01$  ;  $+ \epsilon = 0.04$  ;  $\diamond \epsilon = 0.07$  ;  $\triangle \epsilon = 0.10$  ;  
 $\times \epsilon = 0.13$  ;  $\nabla \epsilon = 0.16$  ;  $\circ \epsilon = 0.19$ ).

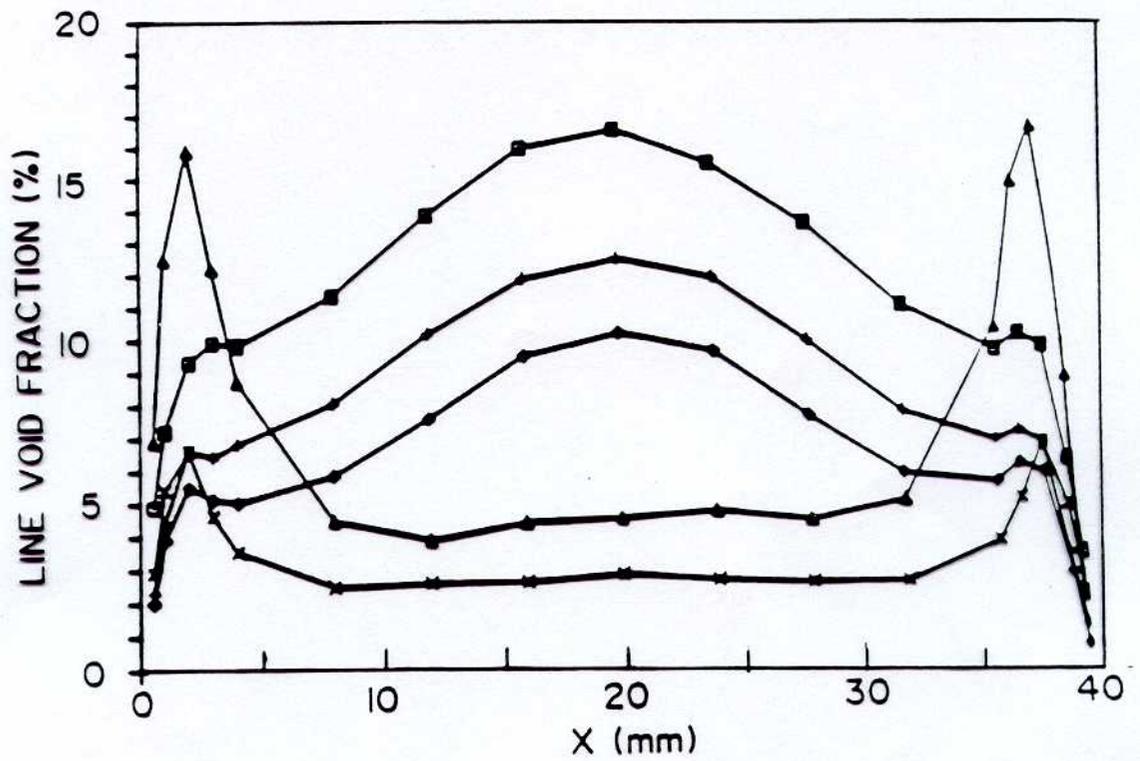


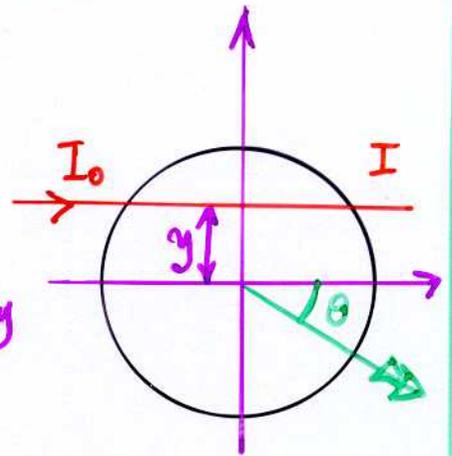
Figure 12 : Void fraction profiles for forced-circulation flow ( $v_l = 2.0$  m/s,  $L = 40$  mm), ( $\times$   $m_g = 0.1$  g/s,  $\epsilon = 0.030$  ;  $\Delta$   $m_g = 0.3$  g/s,  $\epsilon = 0.061$  ;  $\diamond$   $m_g = 0.4$  g/s,  $\epsilon = 0.069$  ;  $+$   $m_g = 0.5$  g/s,  $\epsilon = 0.089$  ;  $\square$   $m_g = 0.7$  g/s,  $\epsilon = 0.123$ ).

# TAUX de VIDE SURFACIQUE.

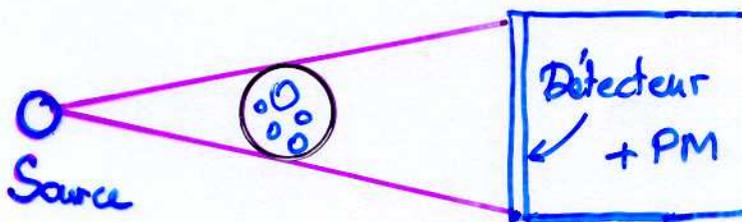
- Atténuation des Rayons X

- Moyenne spatiale  $\overline{R_{G2}}$

$$\overline{R_{G2}} = \frac{1}{\pi R^2} \int_{-R}^R 2 R_{G1}(y) \sqrt{R^2 - y^2} dy$$



- Valeur Instantannée  $R_{G2}$

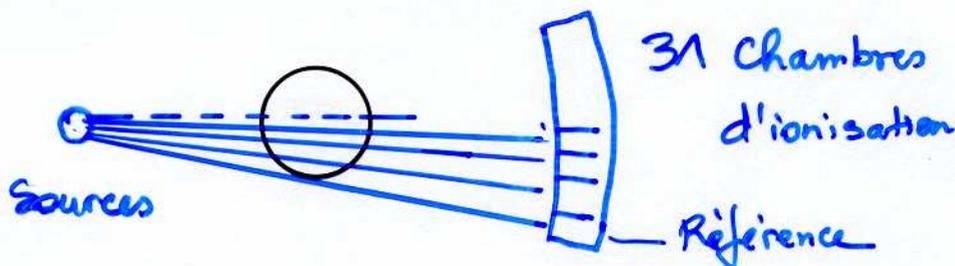


Limitations

$$\Delta R_{G2} \leq \pm 0,05$$

$$0 < R_{G2} < 0,8$$

- Densitomètre multifaisceaux (31)



- Tomographie à Rayons X

on mesure  $\overline{R_{G1}}(y, \theta) \xrightarrow{\text{Inversion}} d_G(r, \theta)$

- Diffusion de neutrons

• traverse l'acier, diffusés + par l'eau.

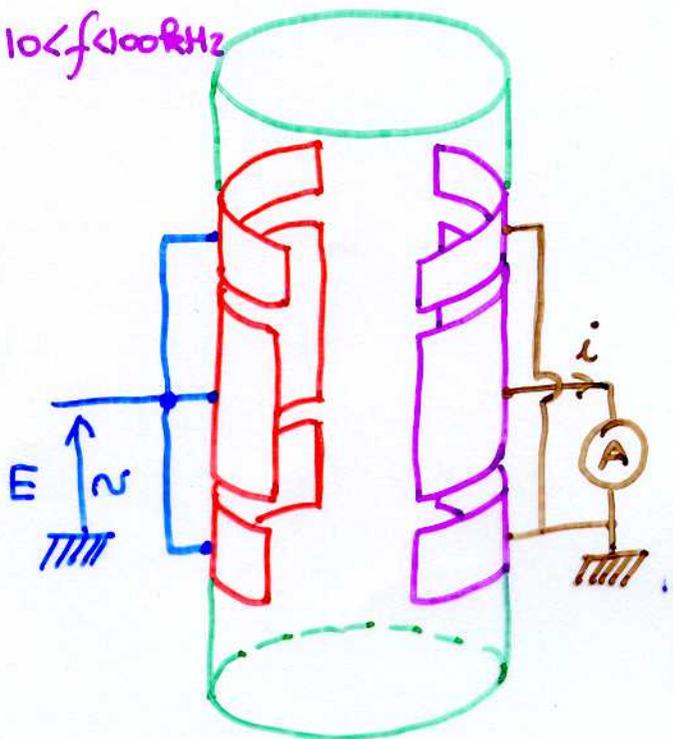
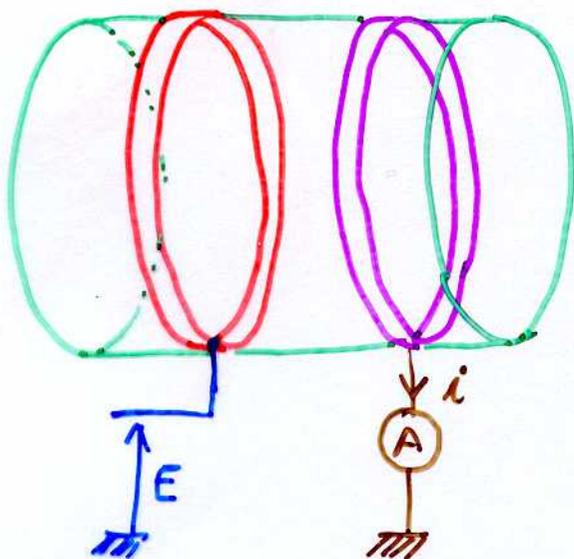
# DENSITOMETRES à IMPEDANCE

- Détermination impédance électrique  $2\phi$

— Résistif :  $\sigma_{2\phi}$

— Capacitif :  $\epsilon_{2\phi}$

$10 < f < 1000 \text{ kHz}$



- Excitation  $E \sim$  signal  $i$

$$I = DE \sigma_c(T, c_1, c_2 \dots) f(R_{G2}, \dots)$$

- $R_{G3} \sim R_{G2}$  evolution lente ;  $R_{G2}(t)$
- Méthode de référence  $I_0 = E \sigma_c(T, c_1, c_2) f_0$
- Etalonnage / Modèle math.  $f(R_{G2}, \dots)$
- Optimisation géométrique  $f(R_{G2}, X)$

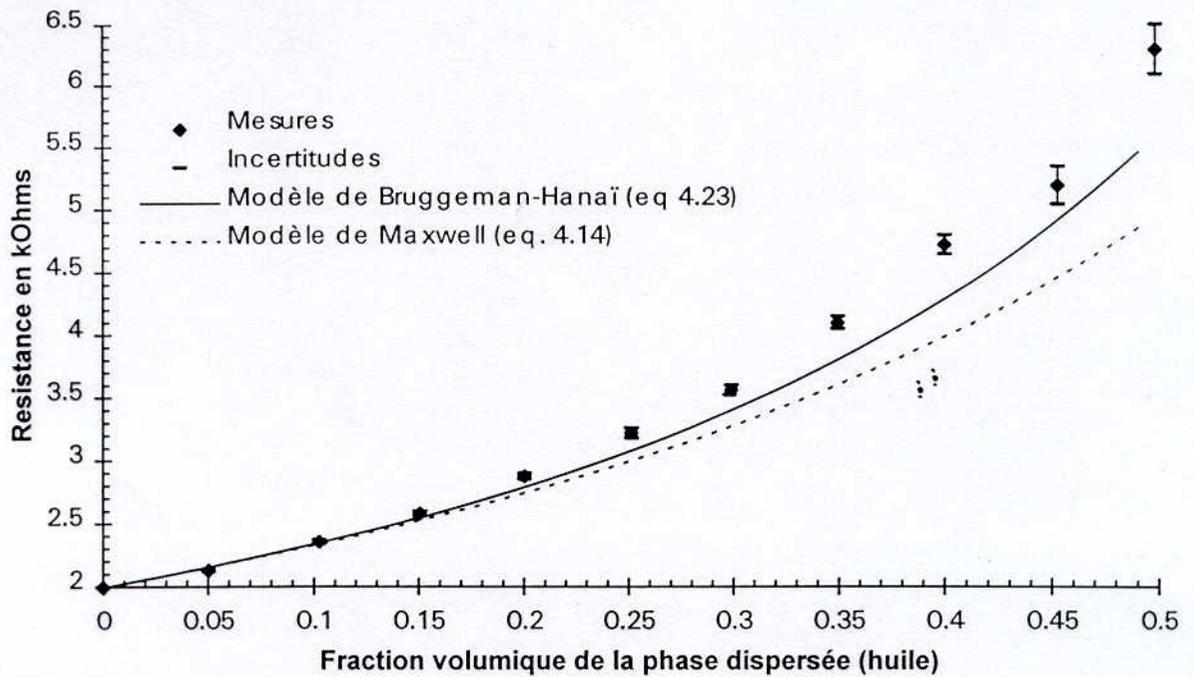


Figure 4.9 : Tracé de la résistance mesurée et calculée pour un écoulement eau-huile, lorsque la vitesse moyenne de l'écoulement est 56,6 cm/s et la température 18 °C ± 0,3.

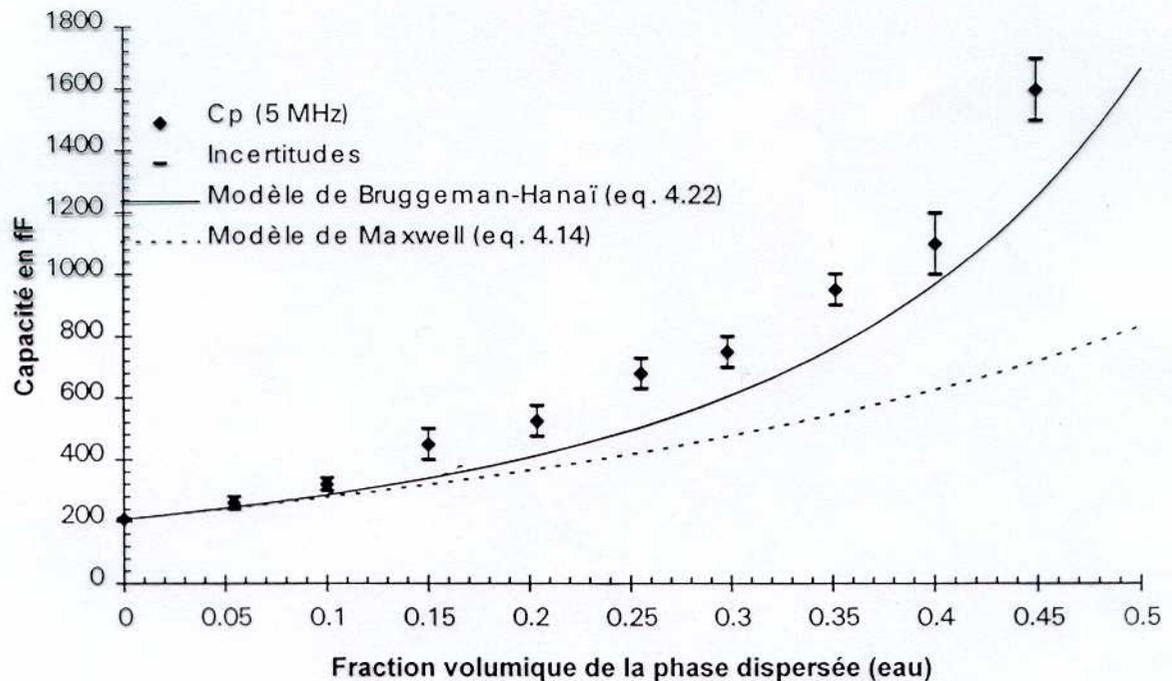
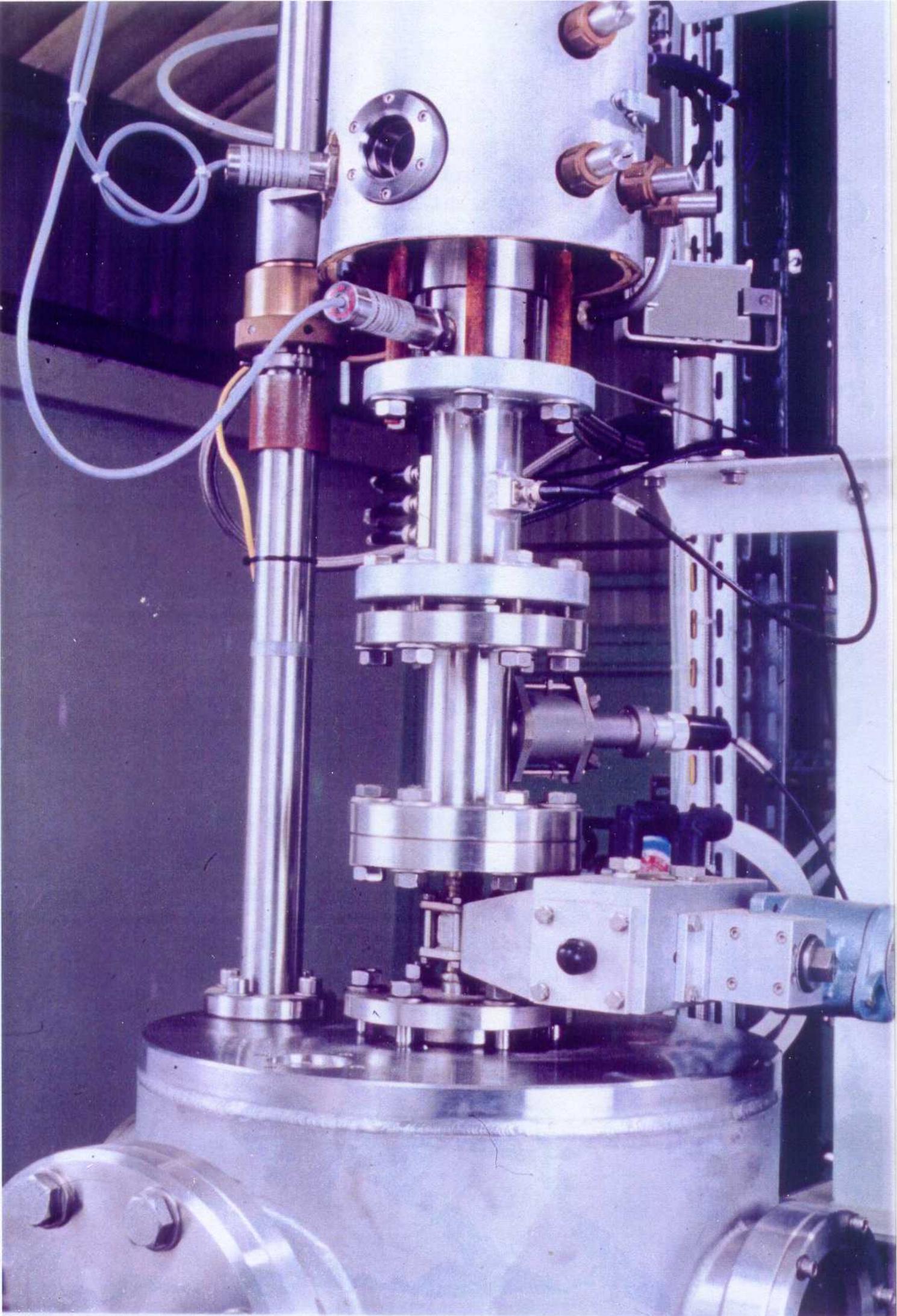
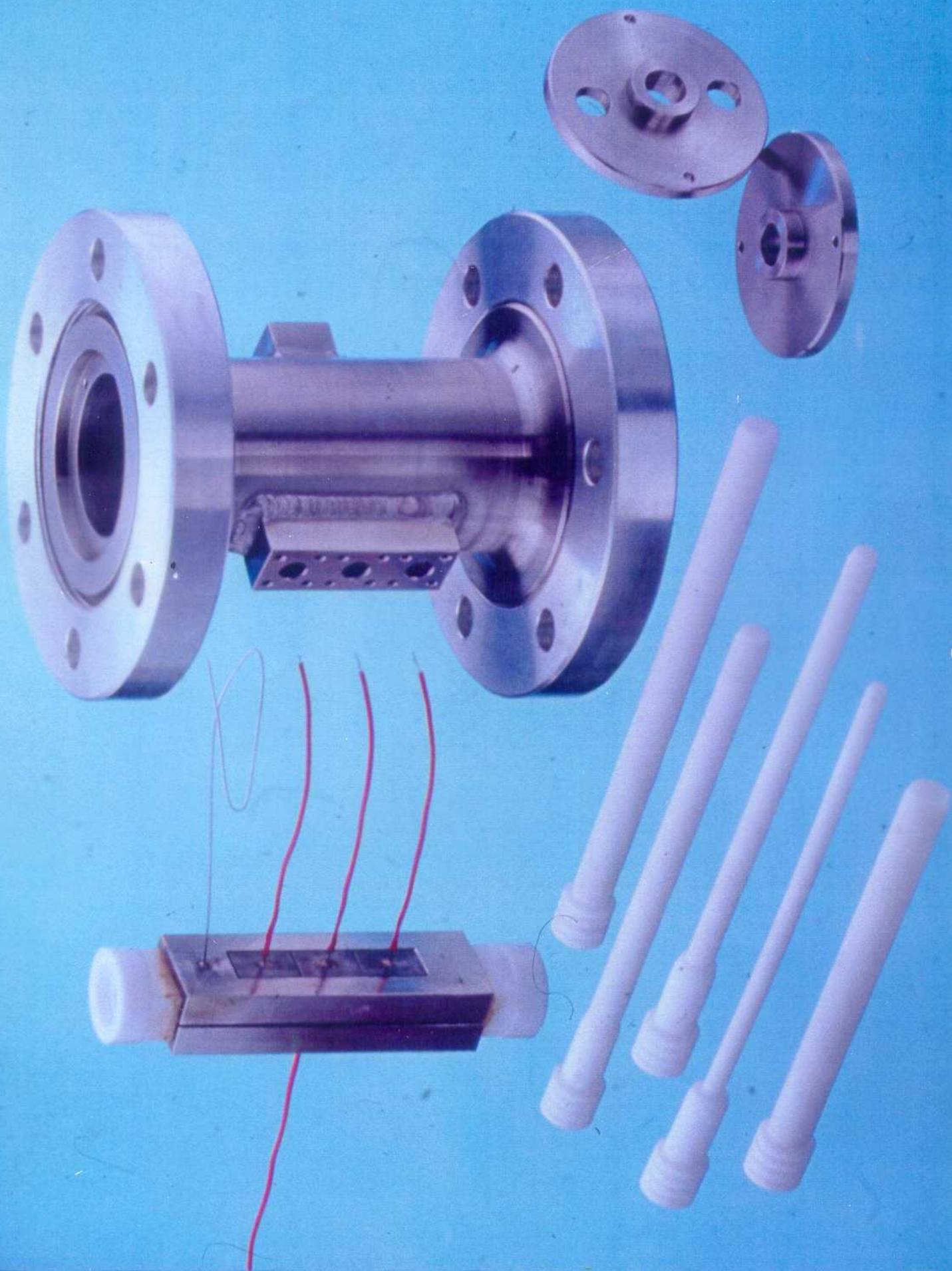


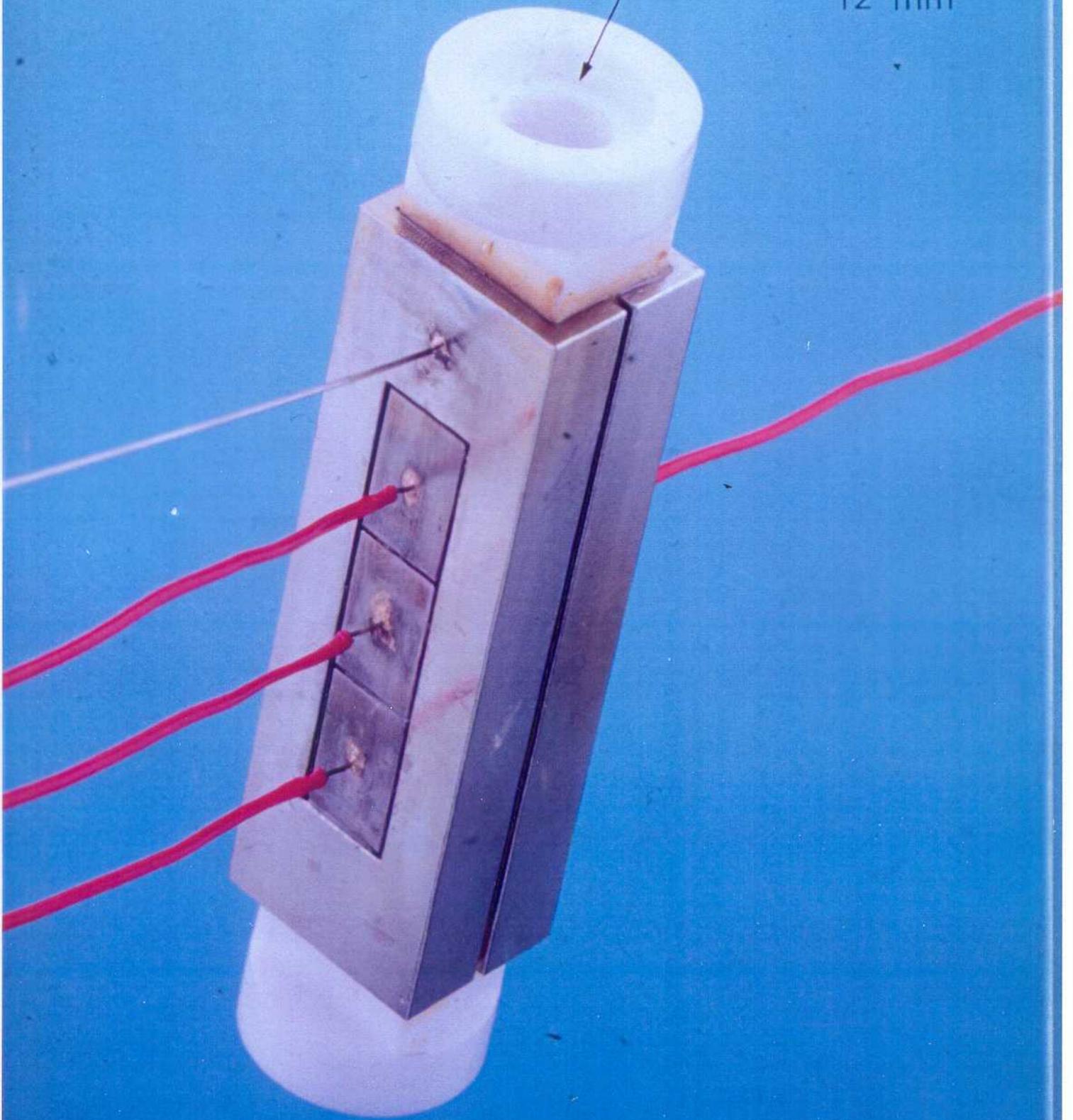
Figure 4.10 : Tracé de la capacité mesurée et calculée en écoulement huile-eau, lorsque la vitesse moyenne est de 56,6 cm/s et la température de 19,5 °C ± 0,1.

d'après Boyer, G. (1996)



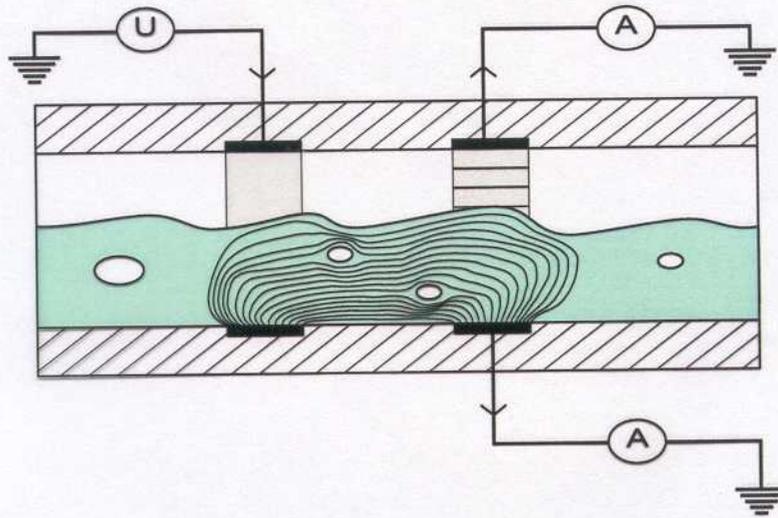


Ø INTERIEUR  
12 mm



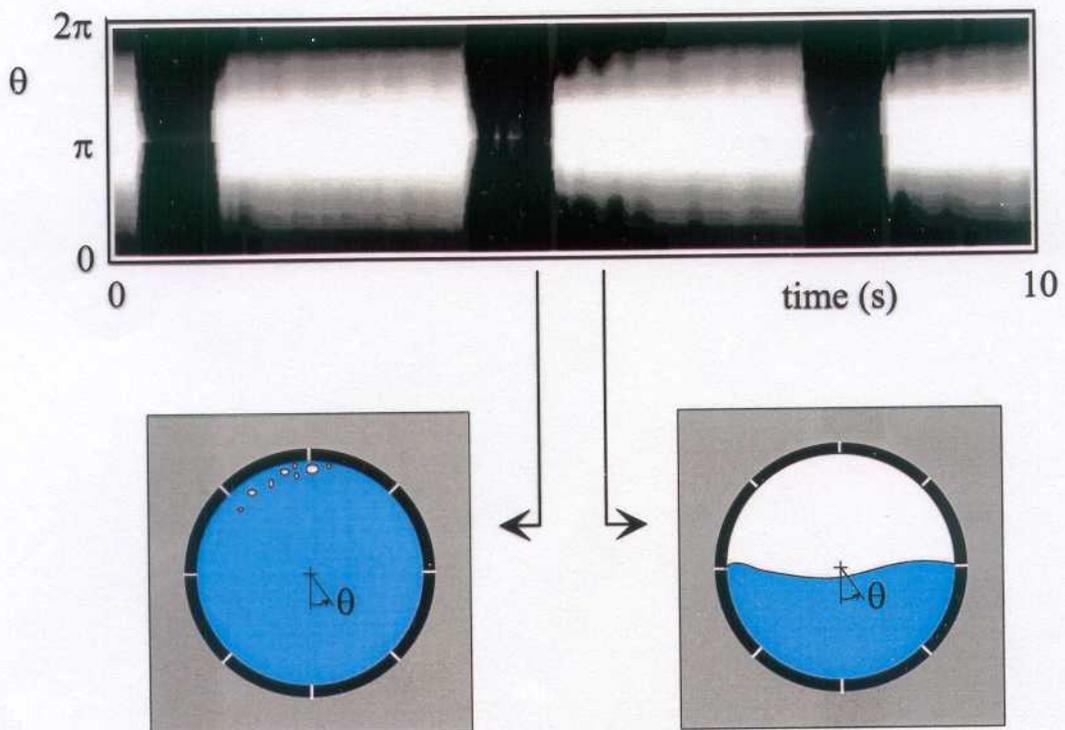
# Impedance probe

- non intrusive
- spatial and dynamic information
- easy optimization (3D simulation)

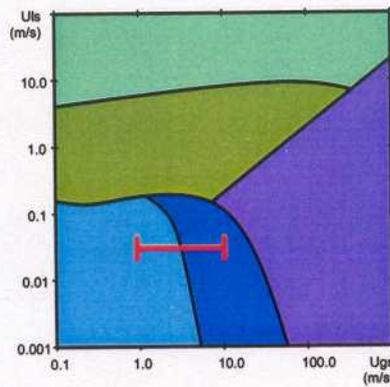


## Flow monitoring (real time)

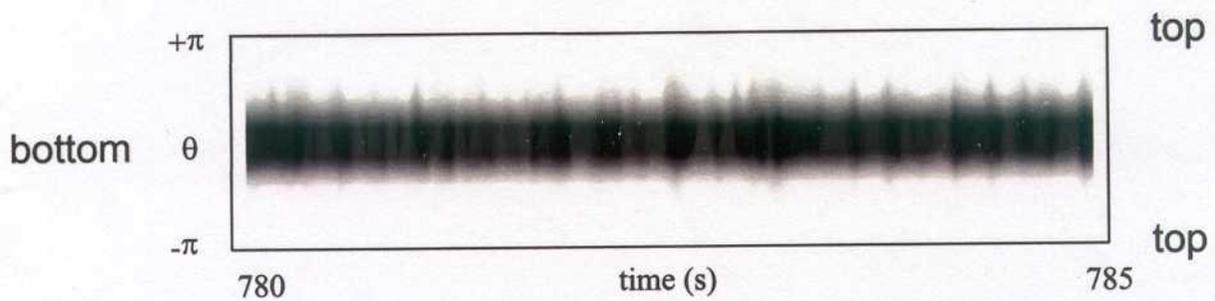
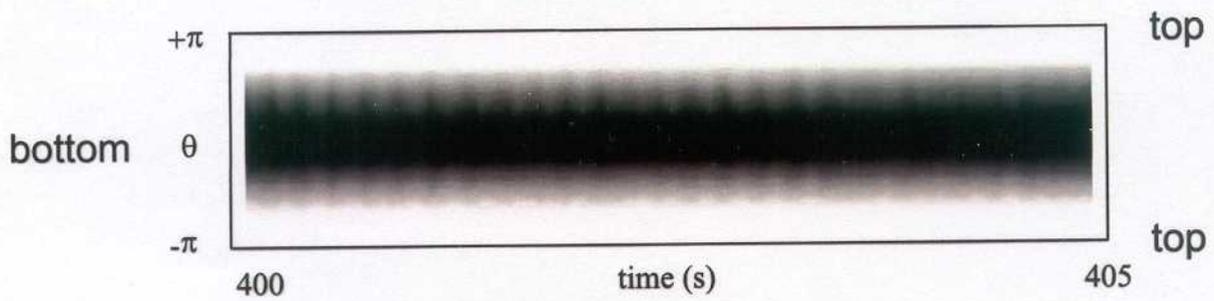
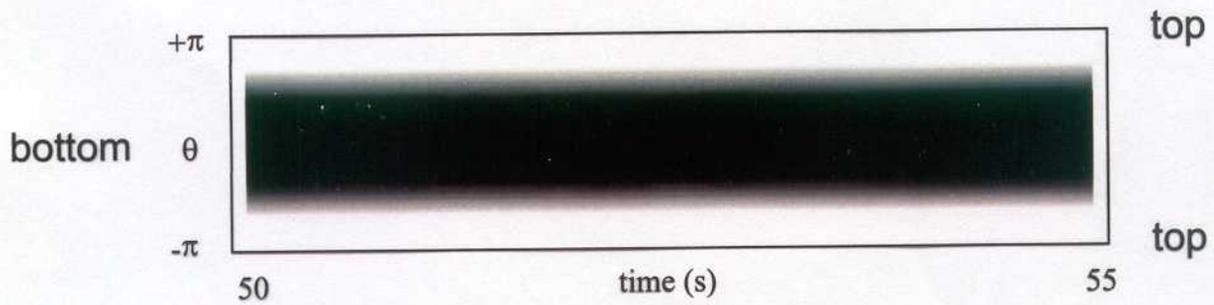
### Slug flow



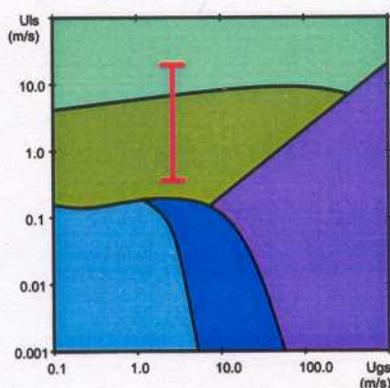
# Smooth - Rugged Stratified Flow Evolution



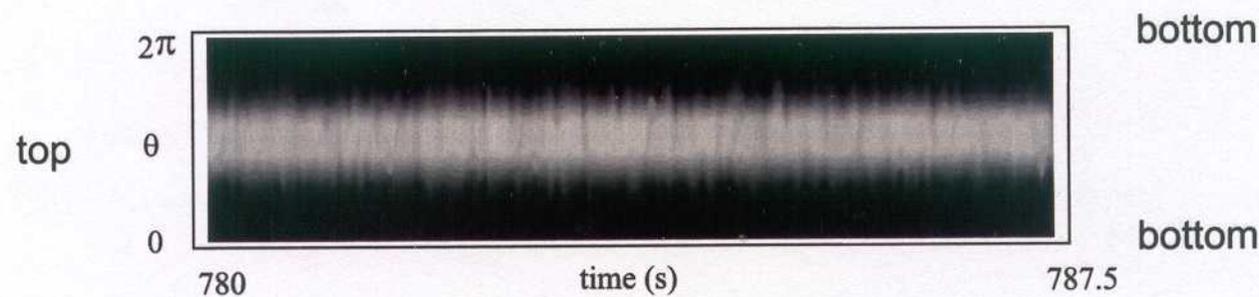
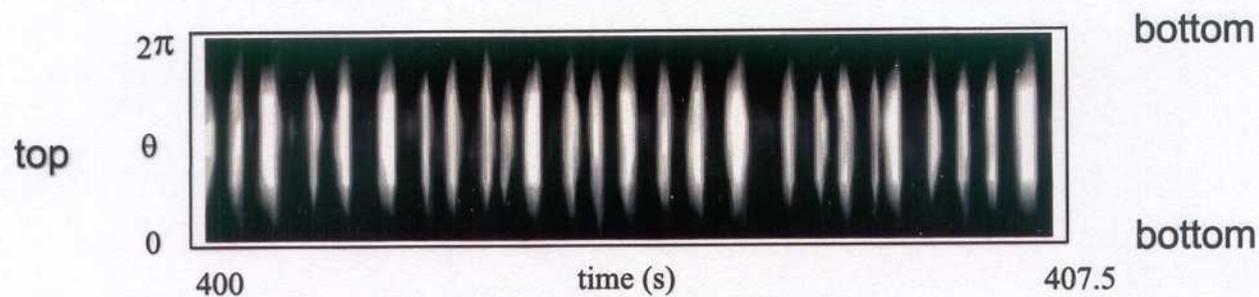
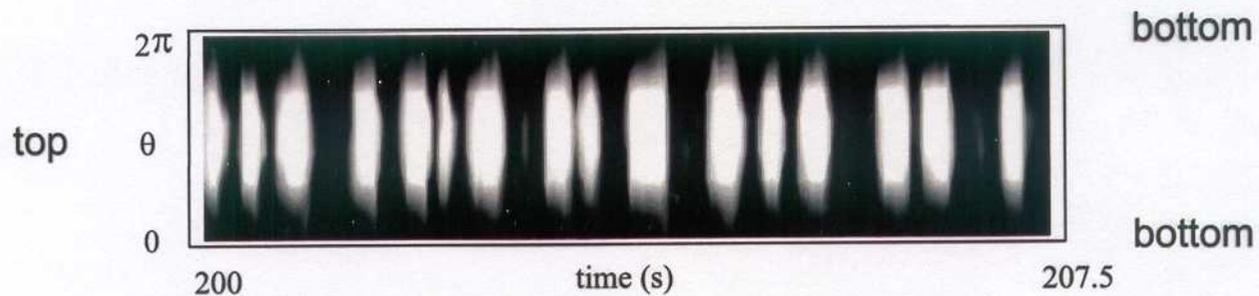
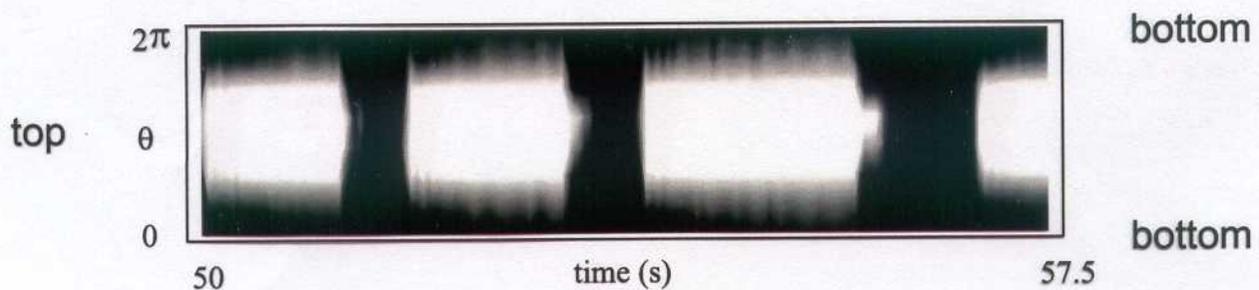
$$Q_{\text{water}} = 0.5 \text{ m}^3/\text{h} - Q_{\text{air}} = 10 - 60 \text{ m}^3/\text{h}$$



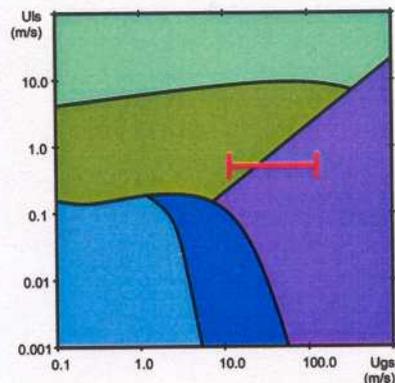
# Intermittent - Bubbly Flow Evolution



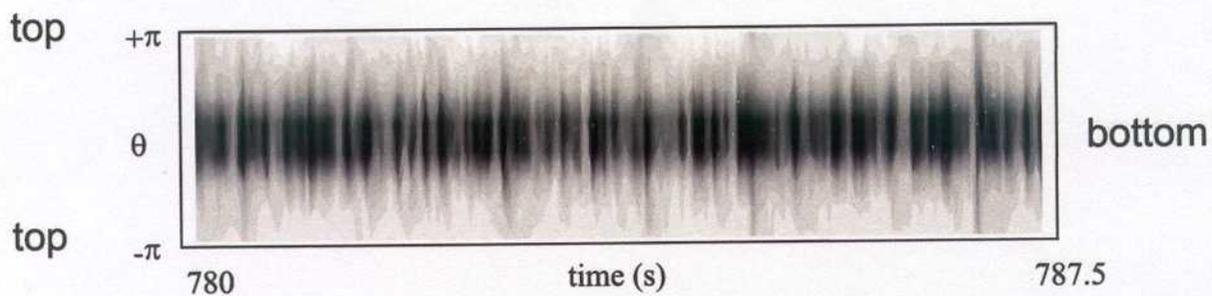
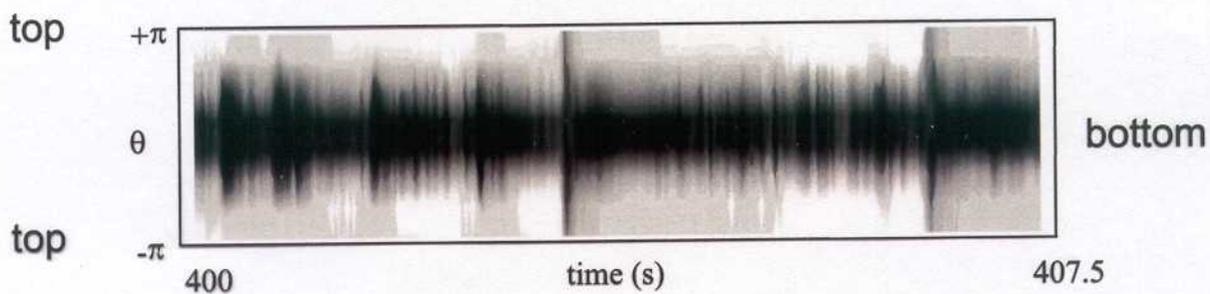
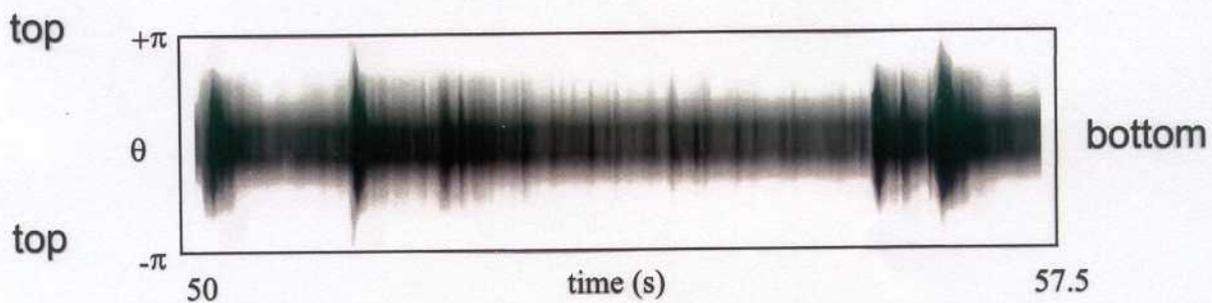
$$Q_{\text{air}} = 7.5 \text{ m}^3/\text{h} - Q_{\text{water}} = 5 - 50 \text{ m}^3/\text{h}$$



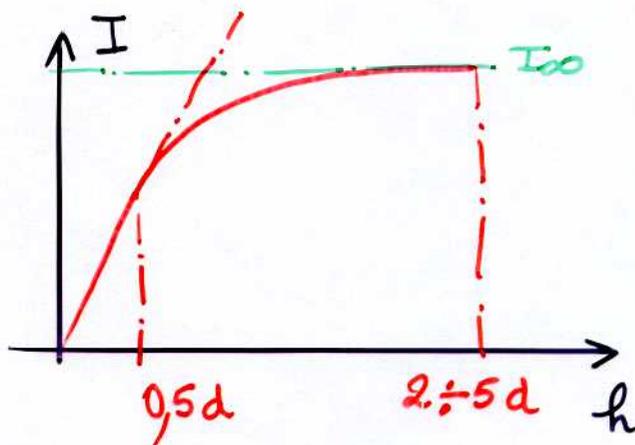
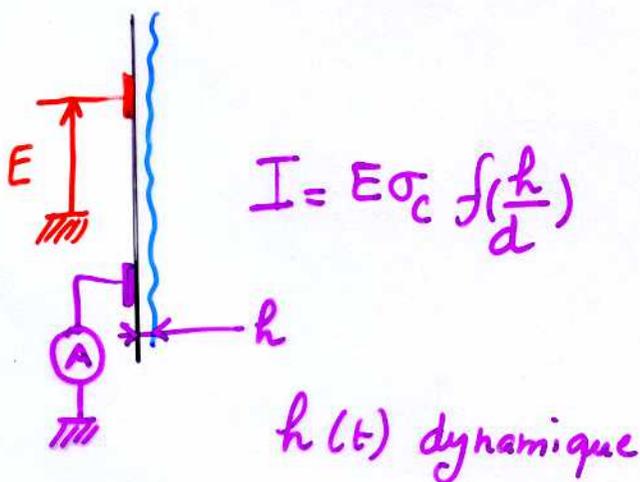
# Intermittent - Annular Flow Evolution



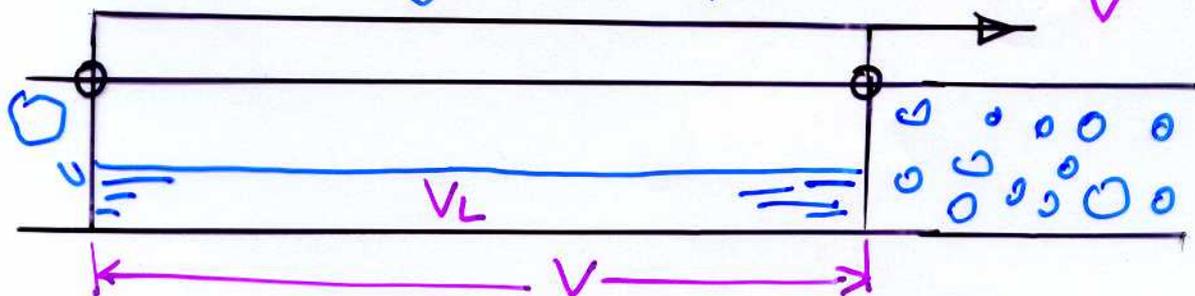
$$Q_{\text{water}} = 15 \text{ m}^3/\text{h} - Q_{\text{air}} = 70 - 250 \text{ m}^3/\text{h}$$



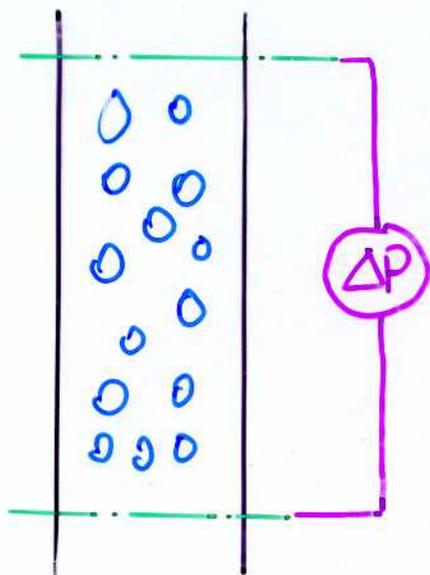
- Epaisseur de film liquide



- Vannes à fermeture rapide :  $R_{L3} = \frac{V_L}{V}$



- Méthode gravitaire



- Vitesses "faibles"  
 $\Rightarrow$  frottement et accélération  $\rightarrow 0$

$$\Delta P = \rho_m g h$$

$$\rho_m = (1 - R_{G3}) \rho_L + R_{G3} \rho_G$$

# LES PREMIERS MODELES SIMPLE de TAUX de VIDE

- Situations d'équilibre (établies)  
transitoires lents.

- Idéalisation de l'écoulement.

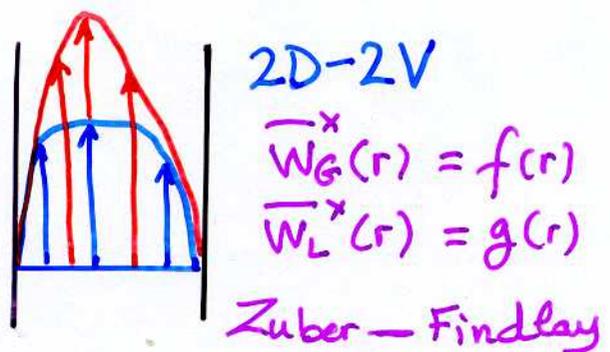
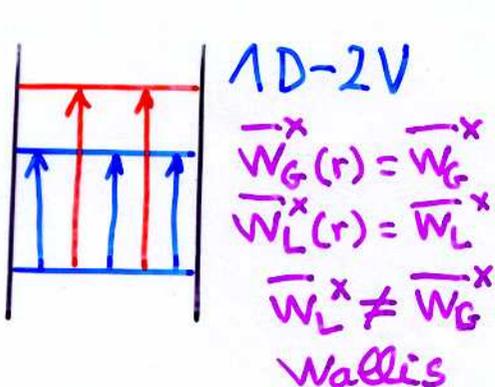
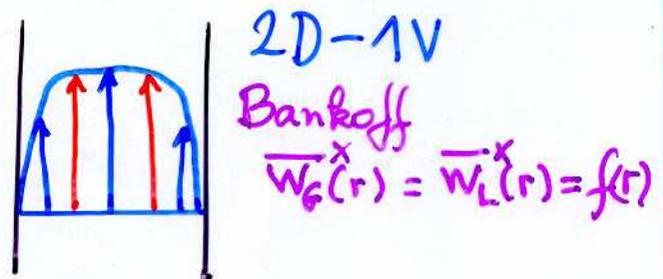
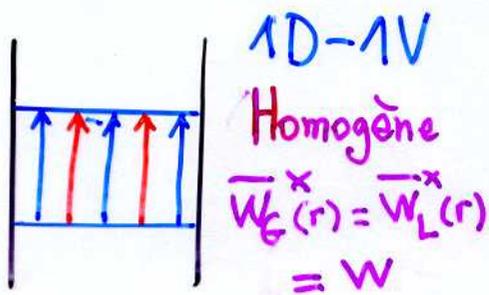
- déséquilibre mécanique  $\overline{w}_G^x \neq \overline{w}_L^x$
- déséquilibre thermodynamique

$$\overline{h}_L^x \neq h_{Lsat}(T)$$

$$\overline{h}_V^x \neq h_{Vsat}(T)$$

- Propriétés géométriques de l'écoulement  
- axisymétrique, profils radiaux...

déséquilibres imposés  $\neq$  modèle non contraint



## LE MODELE HOMOGENE.

• 1D-1V :  $\overline{w_L^x} = \overline{w_G^x} = \text{cste}$

On se donne les débits :  $\overline{Q_G}$  et  $\overline{Q_L}$   
 On calcule :  $\overline{R_G}$  ou  $\overline{\alpha_G}$

Etablissement commun aux 4 modèles

$$\overline{Q_G} \triangleq \int_{A_G} w_G dA \equiv \overline{A_G \langle w_G \rangle} = A \overline{R_G \langle w_G \rangle}$$

(CDM): Commutativité des moyennes spatiales et temporelles

$$\overline{Q_G} = A \overline{R_G \langle w_G \rangle} = A \overline{\alpha_G \overline{w_G^x}}$$

or  $\overline{w_G^x}$  est uniforme ds la section et CDM

$$\overline{Q_G} = A \overline{\alpha_G} \overline{w_G^x} = A \overline{R_G} \overline{w_G^x} \quad \triangle$$

de même pour le liquide :

$$\overline{Q_L} = A (1 - \overline{R_G}) \overline{w_L^x}$$

or les vitesses sont égales

$$\frac{\overline{Q_G}}{\overline{Q_L}} = \frac{\overline{R_G}}{1 - \overline{R_G}} \Rightarrow \overline{R_G} = \frac{Q_G}{Q_L + Q_G} = \frac{Q_G}{Q} \triangleq \beta$$

titre volumique

Modèle homogène

$$\overline{R_G} = \beta$$

## LE MODELE de BANKOFF.

• 2D - 1 V 
$$\begin{cases} \overline{w}_L^x = \overline{w}_G^x = w_c \left( \frac{y}{R} \right)^{\frac{1}{m}} \\ \alpha_G = \alpha_c \left( \frac{y}{R} \right)^{\frac{1}{n}} \end{cases}$$

On se donne les débits:  $\overline{Q}_G$  et  $\overline{Q}_L$

On calcule  $\overline{R}_G \equiv k \alpha^2$

Definition des débits moyens

$$\overline{Q}_G = A k \alpha \overline{w}_G^x = A f(w_c, \alpha_c, m, n)$$

$$\overline{Q}_L = A k (1-\alpha) \overline{w}_L^x = A g(w_c, \alpha_c, m, n)$$

Calcul de

$$k \overline{w}_L^x = k \overline{w}_G^x = h(w_c, m)$$

$$\overline{R}_G = k \alpha^2 = k(\alpha_c, n)$$

on élimine  $\alpha_c$  et  $w_c$

$$R_G = K \beta$$

$$K = \frac{2(m+n+mn)(m+n+2mn)}{(n+1)(2n+1)(m+1)(2m+1)}$$

$$K = 0,6 \div 1 \quad 2 \leq m, n \leq 7$$

Corrélation de Bankoff.

$$K = 0,71 + 0,00145 \mu \quad (\mu \text{ en bar})$$

## MODELE de WALLIS

- 1D-2V  $\begin{cases} \overline{w_L^x} \neq \overline{w_G^x} \text{ mais uniformes} \\ \alpha_G(r) = \alpha_G \text{ "} \end{cases}$

On se donne les débits :  $\overline{Q_L}$  et  $\overline{Q_G}$   
 On cherche :  $\overline{R_G}$

Définition des débits:

$$\boxed{1/A} \quad \overline{Q_G} = \langle \alpha \overline{w_G^x} \rangle = \overline{R_G} \overline{w_G^x}$$

$$\boxed{1/A} \quad \overline{Q_L} = \langle (1-\alpha) \overline{w_L^x} \rangle = (1-\overline{R_G}) \overline{w_L^x}$$

On calcule  $\overline{R_G}$

$$\overline{R_G} = \frac{\overline{Q_G} \overline{w_L^x}}{\overline{Q_L} \overline{w_G^x} + \overline{Q_G} \overline{w_L^x}} = \frac{\beta}{1 + \frac{(1-\overline{R_G})(\overline{w_G^x} - \overline{w_L^x})}{J}}$$

Rappel  $J = \frac{Q}{A} = \frac{Q_L + Q_G}{A}$ .

Exemple : Ecoulement à bulles  $w_{00}$  vit. bulle

$$\overline{w_G^x} - \overline{w_L^x} \approx w_{00} (1 - \overline{R_G})$$

- Corrélation pour  $w_{00}$  ( $D, \sigma, \rho_L, \rho_G, \mu_L$  etc...)  
Clift par exemple
- Diagramme de Wallis

# DIAGRAMME de WALLIS

- Définitions

- $j_k \triangleq \alpha_k \overline{w_k^x} = \overline{x_k w_k}$  ;  $j = j_L + j_G$

vitesse de dérive (vitesse relative / c du vol)

$$v_{kj} \triangleq \overline{w_k^x} - j$$

flux de dérive (flux de phase  $\phi$  / c du volume).

$$j_{GL} \triangleq \alpha_G (\overline{w_G^x} - j)$$

- Hypothèse 1)

$$J_{GL} = \alpha_G j_{GL} = \overline{R_G} (\overline{w_G^x} - j) = (1 - \overline{R_G}) j_G - \overline{R_G} j_L \quad (1)$$

$$j = j_G + j_L = \overline{R_G} \overline{w_G^x} + (1 - \overline{R_G}) \overline{w_L}$$

$$\Rightarrow J_{GL} = \overline{R_G} (1 - \overline{R_G}) (\overline{w_G^x} - \overline{w_L})$$

- Corrélation pour  $\overline{w_G^x} - \overline{w_L}$  en éc à bulles

$$J_{GL} = w_{00} R_G (1 - R_G)^2 \quad (2)$$

- Diagramme de Wallis

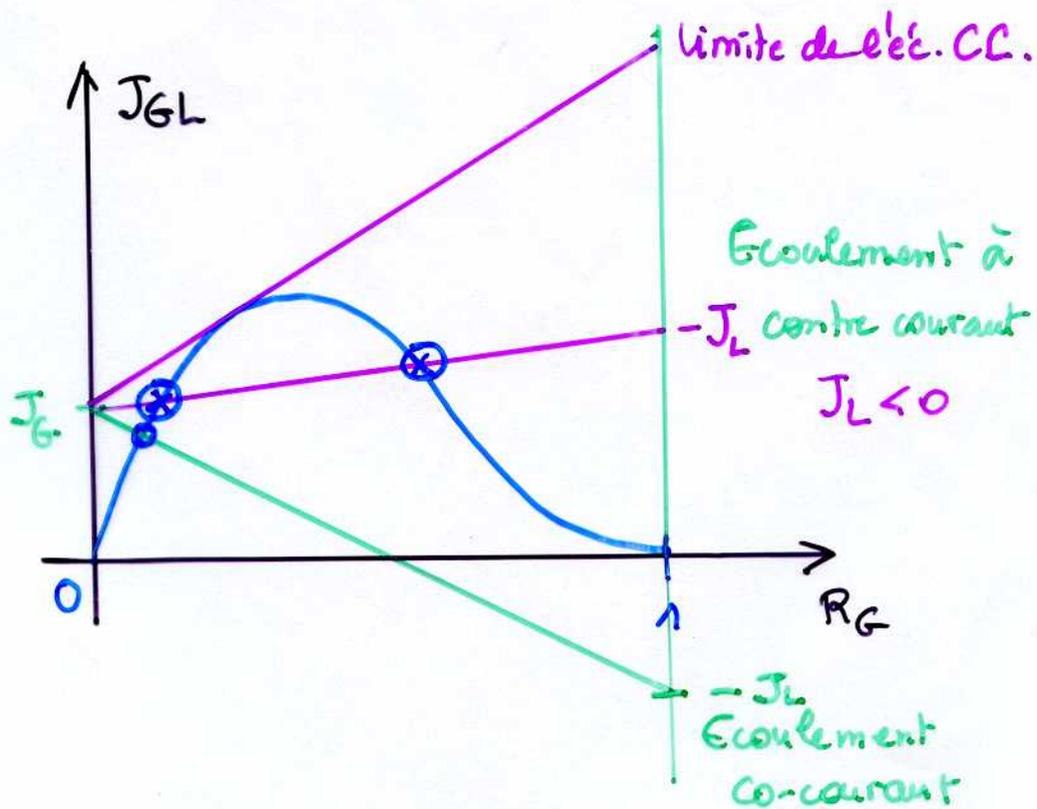
$$J_{GL} = f(R_G)$$

(1) droite

(2) courbe

$R_G = 0$  ;  $j_G$

$R_G = 1$  ;  $-j_L$



- Ecoulement co-courant ascendant.
  - 1 seul régime.  $\odot$
- Ecoulement à contre courant
  - 2 régimes peuvent coexister  $\otimes$

## MODELE de Zuber Findlay

- 2D-2V

- le plus général : on se donne  $\overline{Q}_G$  et  $\overline{Q}_L$   
ou recherche  $\overline{R}_{G2}$

On part de la vitesse de dérive locale

$$\overline{w}_{Gj}^x \triangleq \overline{w}_G^x - j = (1 - \alpha_G)(\overline{w}_G^x - \overline{w}_L^x)$$

On moyenne sur la section le flux de dérive

$$\langle \alpha_G \overline{w}_{Gj}^x \rangle = \langle \alpha_G \overline{w}_G^x \rangle - \langle \alpha_G j \rangle$$

$$\langle \alpha_G \rangle \tilde{w}_{Gj} = J_G - C_0 \langle \alpha_G \rangle \langle j \rangle$$

$$R_G = \frac{J_G}{C_0 J + \tilde{w}_{Gj}}$$

$$\tilde{w}_{Gj} \triangleq \frac{\langle \alpha \overline{w}_{Gj}^x \rangle}{\langle \alpha_G \rangle}$$

$$C_0 = \frac{\langle \alpha_G j \rangle}{\langle \alpha_G \rangle \langle j \rangle}$$

- dérivé sur les autres modèles
- Expériences  $\rightarrow$  Correlations par régime

$$\frac{J_G}{R_G} = FL(J) \Rightarrow C_0 \text{ et } \tilde{w}_{Gj}$$

## Le modèle de Zuber et Findlay

Ce modèle conduit à l'expression suivante du taux de vide :

$$R_G = \frac{\beta}{C_0 + \bar{v}_{gj}/J} \quad ; ;$$

où  $\beta$  est le titre volumique et  $J \hat{=} J_G + J_L$  la vitesse débitante du mélange ;  $C_0$  paramètre de distribution et la vitesse de dérive pondérée  $\bar{v}_{gj}$  dépendent du type d'écoulement.

Les expressions suivantes peuvent être recommandées pour les conduites verticales :

$$C_0 = \left( 1,2 - 0,2 \sqrt{\frac{\rho_G}{\rho_L}} \right) \left( 1 - e^{-18R_G} \right) \quad \text{wall peaking.}$$

Écoulements à bulles :

$$\bar{v}_{gj} = (C_0 - 1)J + 1,4 \left[ \frac{\sigma g(\rho_L - \rho_G)}{\rho_L^2} \right]^{0,25} (1 - R_G)^{1,75}$$

Écoulements à poches :

$$\bar{v}_{gj} = (C_0 - 1)J + 0,35 \left[ \frac{g(\rho_L - \rho_G)D}{\rho_L} \right]^{0,5}$$

Écoulements agités :

$$\bar{v}_{gj} = (C_0 - 1)J + 1,4 \left[ \frac{\sigma g(\rho_L - \rho_G)}{\rho_L^2} \right]^{0,25}$$

Écoulements annulaires :

$$\bar{v}_{gj} = \frac{1 - R_G}{R_G + \left[ \frac{1 + 75(1 - R_G)}{\sqrt{R_G}} \frac{\rho_G}{\rho_L} \right]^{0,5}} \left[ J + \sqrt{\frac{(\rho_L - \rho_G)gD(1 - R_G)}{0,015\rho_L}} \right]$$

MASS FLOW RATE DETERMINATION  
IN TWO-PHASE FLOW;  
THE GRENOBLE EXPERIENCE

- EQUIPMENT
- FLOW RATE CALCULATION

## EQUIPMENT

• Gammadensitometer  $\rightarrow R_{G1}$

• Venturi

$$(1) \quad M_V = k \sqrt{\rho_V \Delta p}$$

$k$  (in  $m^2$ ): coefficient obtained from a calibration in liquid single phase flow

$M_V$  (in kg/s)  $\rho_V$  (in  $kg/m^3$ )  $\Delta p$  (in Pa)

$$(2) \quad \frac{1}{\rho_V} = \frac{x^2}{R_V \rho_G} + \frac{(1-x)^2}{(1-R_V) \rho_L}$$

• Turbine

$$(3) \quad M_T = \lambda \rho_T f$$

$\lambda$  (in  $m^3$ ): coefficient obtained from a calibration in liquid single phase flow

$M_T$  (in kg/s)  $\rho_T$  (in  $kg/m^3$ )  $f$  (in  $s^{-1}$ )

$$(4) \quad \frac{1}{\rho_T} = \frac{x^2}{R_T \rho_G} + \frac{(1-x)^2}{(1-R_T) \rho_L} \quad (\text{Popper, 1961; Rouhani, 1965})$$

## Method # 1 : "Homogeneous model"

### • Equipment :

|                    |   |            |
|--------------------|---|------------|
| Gamma densitometer | → | $R_{G1}$   |
| Venturi            | → | $\Delta p$ |
| Turbine            | → | $f$        |

### • Hypotheses :

- $R_{G2} \equiv R_{G1}$
- $R_V \equiv R_T \equiv R_{G2}$
- Homogeneous model

### • Flow rate calculation : B : (2) & (4) → $\rho_V \equiv \rho_T$

$$\rho_V \equiv \rho_T = R_{G2} \rho_G + (1 - R_{G2}) \rho_L$$

$$(A) \rightarrow \rho_V \equiv \rho_T = R_{G1} \rho_G + (1 - R_{G1}) \rho_L$$

$$(1) \rightarrow M_V \quad (3) \rightarrow M_T$$

$$C \rightarrow \frac{1}{\rho_V} \equiv \frac{1}{\rho_T} \equiv \frac{x}{\rho_G} + \frac{(1-x)}{\rho_L} \rightarrow x$$

### • Consistency check : $M_V \equiv M_T$

### • Application :

Rod bundle blowdown ( $\Omega$ )

## Method #2: "Venturi-turbine"

- Equipment:

Venturi  $\rightarrow \Delta p$

Turbine  $\rightarrow f$

- Hypotheses:

A.  $\rho_V \equiv \rho_T \hat{=} \rho$

- Flow rate calculation:

$$M_V \equiv M_T \hat{=} M$$

$$(1) \& (3) \rightarrow M = \frac{R^2 \Delta p}{\lambda f}$$

If B. Homogeneous model  $\rightarrow \frac{1}{\rho} = \frac{x}{\rho_G} + \frac{(1-x)}{\rho_L}$

- Application:

Rod bundle blowdown ( $\Omega$ )

## TURBINE FLOWMETERS

DIAMETER :

12 TO 154 MM

BLADE MATERIAL :

STAINLESS STEEL  
TITANIUM

MAXIMUM ROTATIONAL SPEED :

12 MM DIA : 120 000 RPM  
154 MM DIA : 15 000 RPM

MAXIMUM ROTATIONAL ACCELERATION :

600 000 REVOLUTIONS/S<sup>2</sup>

MAXIMUM OPERATING TEMPERATURE :

780°C  
900°C FOR SHORT PERIODS

DYNAMIC RANGE :

0.05 TO 225 M/S (WATER ; 46 MM DIA)

|      |    |  |
|------|----|--|
| WASH | 36 | Measurement of the INTEGRAL specific area $\Gamma(t)$<br>in pipe flows |
|      | 26 |  |

Methods compared:

- photographic (bubble diameter measurements)
- chemical (oxidation of sodium sulfite into sodium sulfate)
- optical (light attenuation)

|      |    |                     |
|------|----|---------------------|
| WASH | 36 | Photographic method |
|      | 27 |                     |

- Advantages:
  - instantaneous picture
- Disadvantages:
  - limited to bubbly or droplet flow
  - 2D picture of the flow near the wall
  - very tedious

|      |    |                 |
|------|----|-----------------|
| WASH | 36 | Chemical method |
|      | 28 |                 |

- Advantages:
  - does not depend on flow pattern
- Disadvantages:
  - limited to steady state flow
  - limited to well defined contactors
  - requires special fluids (aqueous solution of sodium sulfite, oxygen and  $Co^{++}$  as a catalyst)
  - very lengthy (concentration determination)
  - requires a calibration set-up

|      |    |                   |
|------|----|-------------------|
| WASH | 36 | Optical technique |
|      | 31 |                   |

• Advantages:

- simple
- gives instantaneous values

• Disadvantages:

- limited to bubbly or droplet flow
- limited to homogeneous dispersions

$$\Gamma = \frac{4}{d} \ln \frac{I_0}{I_m}$$

LED: light emitting diode

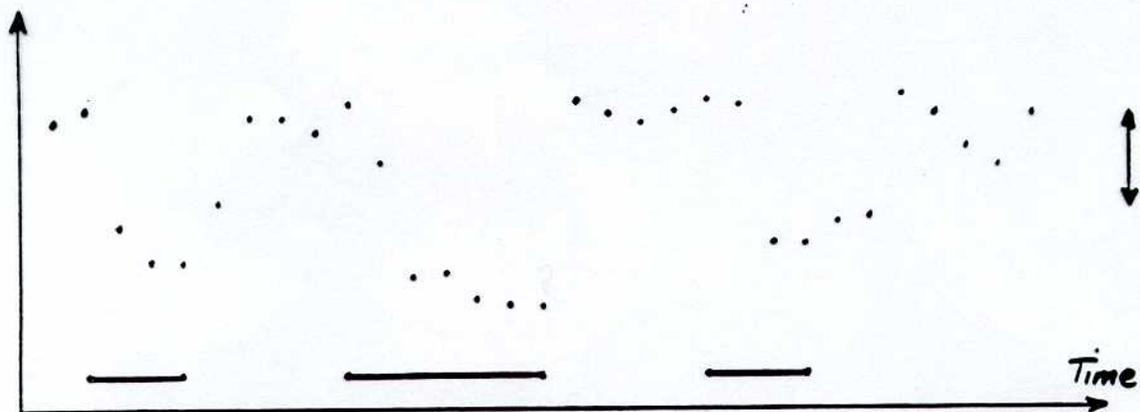
PR: photo receiver

$I_0$ : emitted intensity

$I_m$ : received intensity

## THERMAL ANEMOMETER SIGNAL PROCESSING

### • AMPLITUDE DISCRIMINATION: RESCH

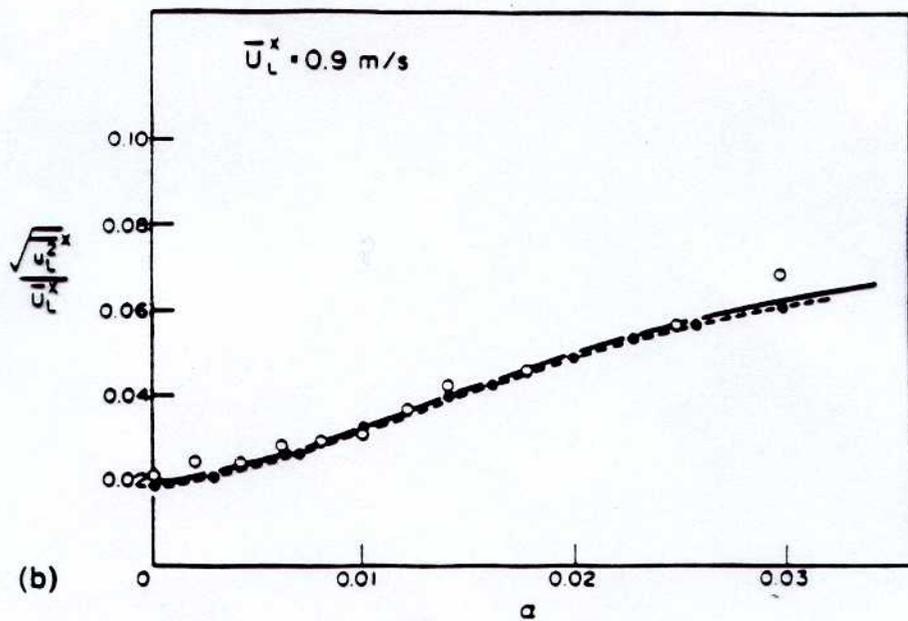
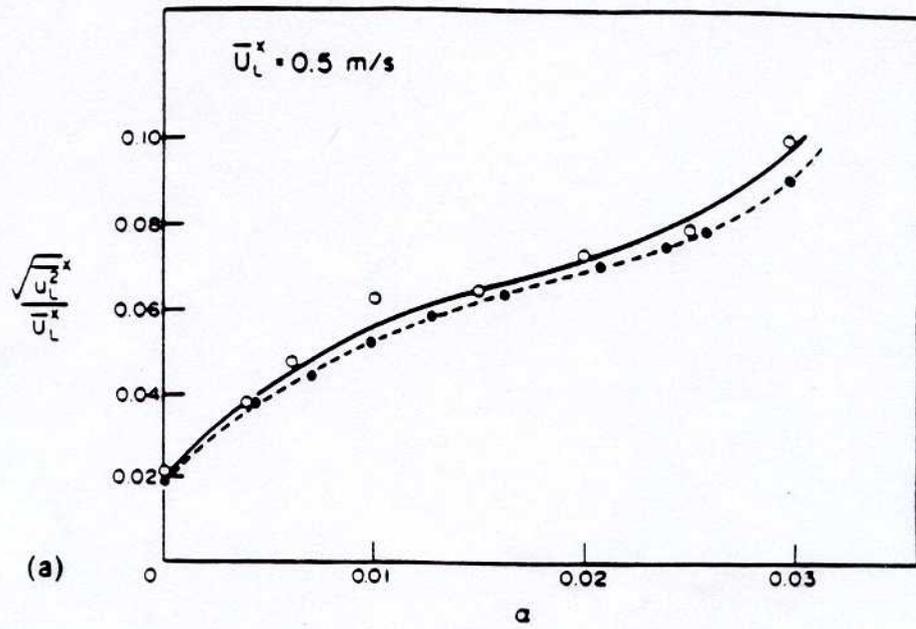


### • OTHER TECHNIQUES :

- PDF splitting (DELHAYE, GALAUP, HERRINGE)
- Threshold on the signal (JONES)
- Threshold on the signal derivative (DUKLER)

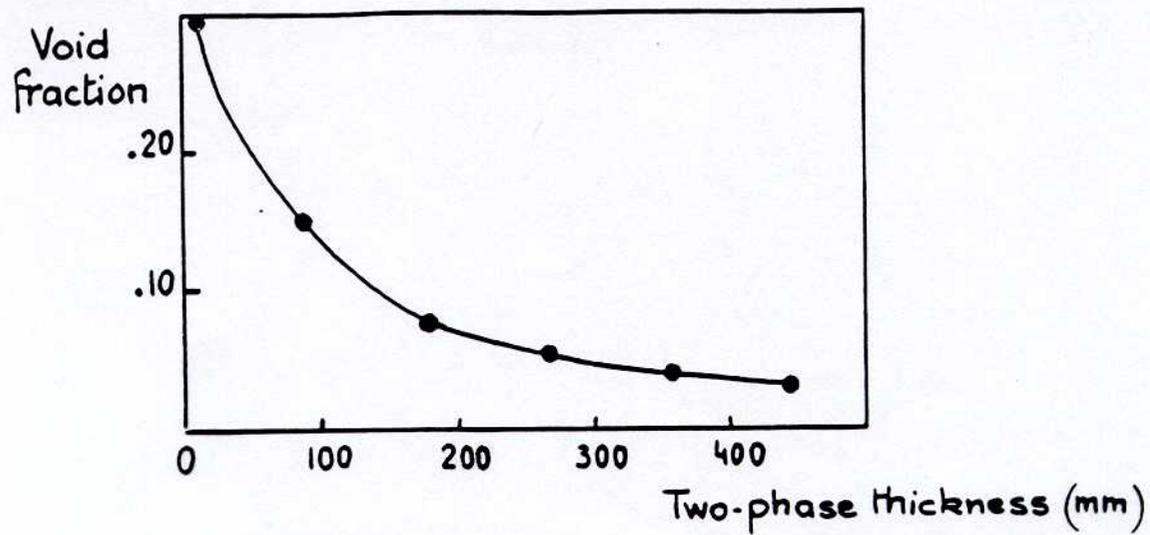
### • GIVES :

- Local void fraction  $\alpha_g$
- Instantaneous liquid velocity  $w_L$



MARIÉ, 1983

## Limitations



See Marié & Lance (1984)