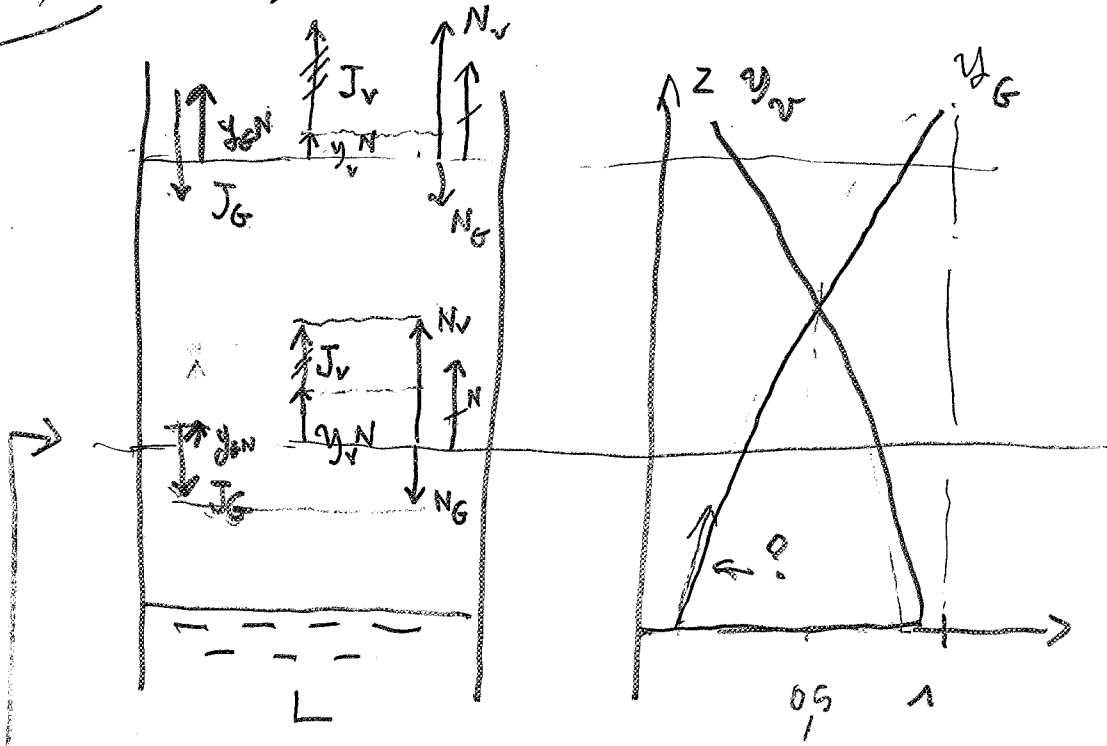


# Compte et S JDM

- 1 -



$$pV = n \tilde{R} T ; \quad p = \frac{n}{V} \tilde{R} T = c \tilde{R} T$$

$p$  &  $T$  est alors  $c$  est constant  $\rightarrow$  Unif. non caisses.

$$c \stackrel{\Delta}{=} c_G + c_V$$

$$c_V v_V + c_G v_G \stackrel{\Delta}{=} cV$$

$$N_V = c_V v_V = c_V V + \underbrace{c_V (v_V - V)}_{J_V} \quad \nabla \cdot N_V = 0$$

$$N_G = c_G v_G = c_G V + \underbrace{c_G (v_G - V)}_{J_G} \quad \nabla \cdot N_G = 0$$

$$J_V + J_G = 0$$

$$N_G = \text{cte} ; \quad N_V = \text{cte}$$

$$N = N_V + N_G = \text{cte}$$

$$N_V = y_V N + J_V > 0$$

$$N_G = y_G N + J_G < 0$$

$N_V$  et  $N_G$  dit par ses cds d'eq. thermo.

Melanges -  
constants.

Vapeur / liquide, Raoult  $p_V = p_{\text{sat},V}(T_V) x_V \quad x_V \approx 1$

ou

gaz / liquide, Henry  $p_G = H(T_G) x_G \quad x_G \ll 1$

$\Delta P_{SL}$

$w_{20} = H_{20}$

ou

$$y_{vo} = \frac{p_{satv}(T)}{p} = \frac{p_v}{p}$$

je me donne  
le compo  
du liquide

$$y_{Go} = \frac{p_G}{p} = \frac{H(T) x_G}{p} = \frac{p_G}{p}$$

$$J_v = N_{vr} - yN = -CD_{vG} \frac{dy_v}{dz}$$

$$\int_{y_{vo}}^{y_{vL}} \frac{dy_v}{y_v N - N_{vr}} = \left( \frac{1}{CD_{vG}} \right) \int_0^L dz$$

$$\frac{1}{N} \ln \left( \frac{y_{vL} N - N_{vr}}{y_{vo} N - N_{vr}} \right) = \frac{L}{CD_{vG}}$$

$$N = \frac{CD_{vG}}{L} \ln \frac{y_{vL} - N_{vr}/N}{y_{vo} - N_{vr}/N} \quad \downarrow + y_{vo} - y_{vL}$$

$$N = \frac{CD_{vG}}{L} \ln \left( 1 + \frac{y_{vL} - y_{vo}}{y_{vo} - N_{vr}/N} \right)$$

$N_{vr}/N$  may be  $\neq 1$

$$T = 20^\circ\text{C} \quad p = 1 \text{ atm}$$

Example

Perry Table 2-122 p. 2-125  $H = 6,64 \cdot 10^{-4} \text{ atm}$   
 — 2-5 p. 2, 49  $p_{\text{sat}}(20) = 17,535 \text{ mm Hg}$   
 $1 \text{ atm} \equiv 760 \text{ mm Hg}$

$$\frac{p_{\text{sat},v}}{p} = \frac{17,535}{760} = 2,307 \cdot 10^{-2} \quad (\approx 23 \text{ mbar})$$

$$\frac{H(T)}{p} = 6,64 \cdot 10^{-4}$$

$$\left\{ \begin{array}{l} y_v = 2,307 \cdot 10^{-2} x_v \\ y_G = 6,64 \cdot 10^{-4} x_G \\ y_v + y_G = 1 \\ x_G + x_v = 1 \end{array} \right. \quad \left\{ \begin{array}{l} \bar{\alpha} 80^\circ\text{C} \quad H = 10,7 \cdot 10^{-4} \\ p_{\text{sat}}(80^\circ\text{C}) = 355,1 \text{ mmHg} \end{array} \right.$$

$$y_v = 2,307 \cdot 10^{-2} x_v$$

$$(1 - y_v) = 6,64 \cdot 10^{-4} (1 - x_v)$$

$$y_v = a x_v \quad (1 - y_v) = b (1 - x_v) \quad 1 - b = y_v - b x_v$$

$$\left\{ \begin{array}{l} y_v - b x_v = 1 - b \\ y_v - a x_v = 0 \end{array} \right.$$

$$+ a x_v - b x_v = 1 - b$$

$$x_v = \frac{1-b}{a-b} \quad x_G = \frac{a-b-1+b}{a-b} = \frac{a-1}{a-b}$$

$$y_v = \frac{a(1-b)}{a-b} \quad y_G =$$

Dans les situations où  $\rho D$  est constant.  
Analyse en masse. (ex beaucoup d'air)

$$n = \frac{\rho D v_G}{L} \ln \left( 1 + \frac{\omega_{VL} - \omega_{V0}}{\omega_{V0} - n_v/n} \right)$$

ds la pratique  $p(T, \omega)$  etc avec  $\rightarrow$  simplifiés.

$$n = \frac{\rho D v_G}{L} \ln (1 + B_V)$$

Cas d'une seule espèce binaire

$$B_V \triangleq \frac{\omega_L - \omega_{V0}}{\omega_{V0} - n_v/n} = \frac{\omega_L - \omega_{V0}}{\omega_{V0} - 1} = \frac{\omega_{L0} - \omega_i}{\omega_i}$$

ice  
↓

$B_V$  = nombre de transfert de masse

$$j_v = n_v - \omega_v n = - \rho D v_G \frac{d\omega_v}{dz}$$

$$\int_{\omega_{V0}}^{\omega_v(z)} \frac{d\omega_v}{\omega_v n - n_v} = \frac{1}{\rho D v_G} \int_0^z dz$$

↑  
valeur moyenne sur le film

$$\frac{1}{h} \ln \left( \frac{\omega_v n - n_v}{\omega_{v0} n - n_v} \right) = \frac{z}{\rho D_{vg}}$$

$$\frac{1}{L} \ln \left( \frac{\omega_{vL} n - n_v}{\omega_{v0} n - n_v} \right) = \frac{nz}{\rho D_{vg}}$$

$$\ln \left( \frac{\omega_v n - n_v}{\omega_{v0} n - n_v} \right) = \frac{nz}{\rho D_{vg}} = \frac{z}{L} \ln \left( \frac{\omega_{vL} n - n_v}{\omega_{v0} n - n_v} \right) \Rightarrow$$

def du coefficient de T & M

$$\left[ \begin{array}{c} \left( \frac{\omega_v - n/n_v}{\omega_{v0} - n/n_v} \right) \\ \left( \frac{\omega_{vL} - n/n_v}{\omega_{v0} - n/n_v} \right)^{\frac{z}{L}} \end{array} \right]$$

$$j_{m0} \stackrel{\Delta}{=} g_{mv} (\omega_{v0} - \omega_{vL})$$

↑ massique

masse  
débit  
be  
moyen



flux inf ≠ flux total

En général  $\Delta$  (surface extérieure, basin.)

$$j_{i,s} \stackrel{\Delta}{=} g_{mi} (\omega_{is} - \omega_{ie})$$

flux total :  $n_{is} = \omega_{is} \dot{m}_s + j_{is} = \omega_{is} \dot{m}_s + g_{mi} (\omega_{is} - \omega_{ie}) \dot{m}_s$

div/n =  $\dot{m}$

$$\frac{n_{is}}{\dot{m}} = \omega_{is} + \frac{g_{mi} (\omega_{is} - \omega_{ie})}{\dot{m}}$$

$$g_{mi} \frac{w_{ie} - w_{is}}{\dot{m}} = w_{is} - \frac{n_{is}}{\dot{m}}$$

$\Delta n$  ← flux total

$$\dot{m} = g_{mi} \left( \frac{w_{ie} - w_{is}}{w_{is} - \frac{n_{is}}{\dot{m}}} \right)$$

$\dot{m}$   
calculé  
n'importe  
quelle  
manière

$$w_{ie} = \frac{n_{is}}{\dot{m}}$$

$$\dot{m} = g_{mi} B_{mi}$$

paramètre moteur du TdM

TdM en terme de flux total -

- mélange binaire  $g_{m1} = g_{m2}; B_{m1} = B_{m2}$

- 1 seule espèce transmise  $v; F$  inerte.

$$B_{mv} = \frac{w_{vL} - w_{v0}}{w_{v0} - 1} = \frac{w_{G0} - w_{GL}}{w_{G0}}$$

# Analyse au régime stat Stefan en masse

$$n = \frac{\rho D_{vg}}{L} \ln \left( 1 + \frac{w_{vL} - w_{v0}}{w_{v0} - n_v/n} \right)$$

$$n = \frac{\rho D_{vg}}{L} \ln (1 + B_{mv})$$

1)  $|B_{mv}| \ll 1$  transfert diffusif régulier.  
faibles TAM

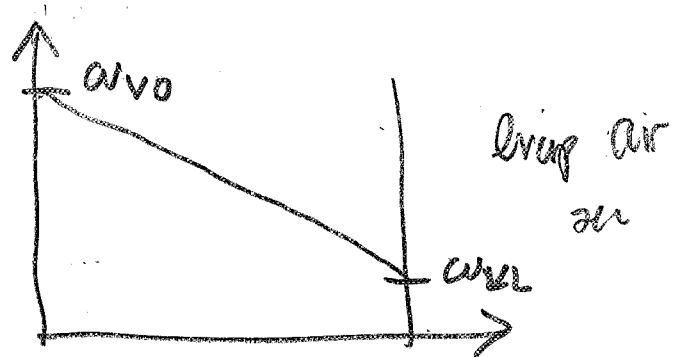
$$\dot{m} \approx \frac{\rho D_{vg}}{L} B_{mv} = \underbrace{g_{mv}^*}_{\frac{a_{mv}}{g_{mv}^0}} B_{mv}$$

$$n_v = a_v/n - \rho D_{vg} \frac{dw_v}{dz}$$

profil linéaire

analyse conduction

$$\frac{d}{dz} n_v \approx -\rho D_{vg} \frac{d^2 w_v}{dz^2} = 0$$



$$B_{mv} = \frac{\omega_{vl} - \omega_i}{\omega_i}$$

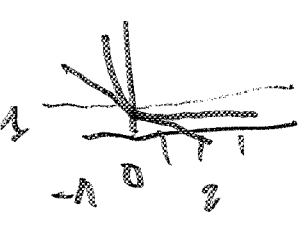
2)  $B_{mv} < 0$

$$-1 < B_{mv} < 0$$

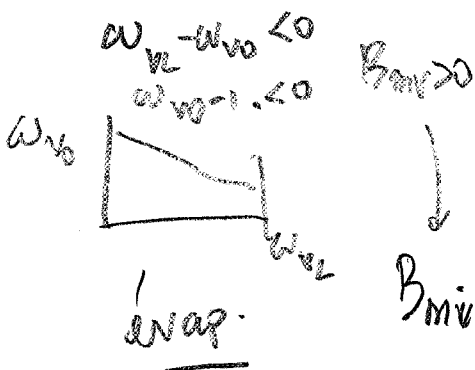
$$0 < B_{mv} < \infty$$

$$\eta = \frac{\rho D v_g}{L} \cdot \ln(1 + B_{mv}) = g_{dmv} B_{mv}$$

Blowing factor



$$g_{dmv} = \frac{\rho D v_g}{L} \frac{\ln(1 + B_{mv})}{B_{mv}} = g_{dmv}^* \frac{\ln(1 + B_{mv})}{B_{mv}}$$

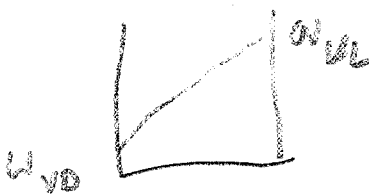


$$B_{mi} > 0 \Rightarrow g_{dmv}^* \left( 1 - \frac{B_{mv}}{2} + O(B_{mv}^2) \right)$$

$B_{mi} > 0 \quad h > 0$  mass away from the wall

$$g_{dmv} < g_{dmv}^* ; \quad 0 < \frac{\ln(1 + B_{mi})}{B_{mi}} < 1$$

condensation  $B_{mi} < 0 \quad h < 0$  suction, flow towards the wall



$$1 < \frac{\ln(1 + B_{mi})}{B_{mi}} < \infty$$

$$g_{dmv} > g_{dmv}^*$$

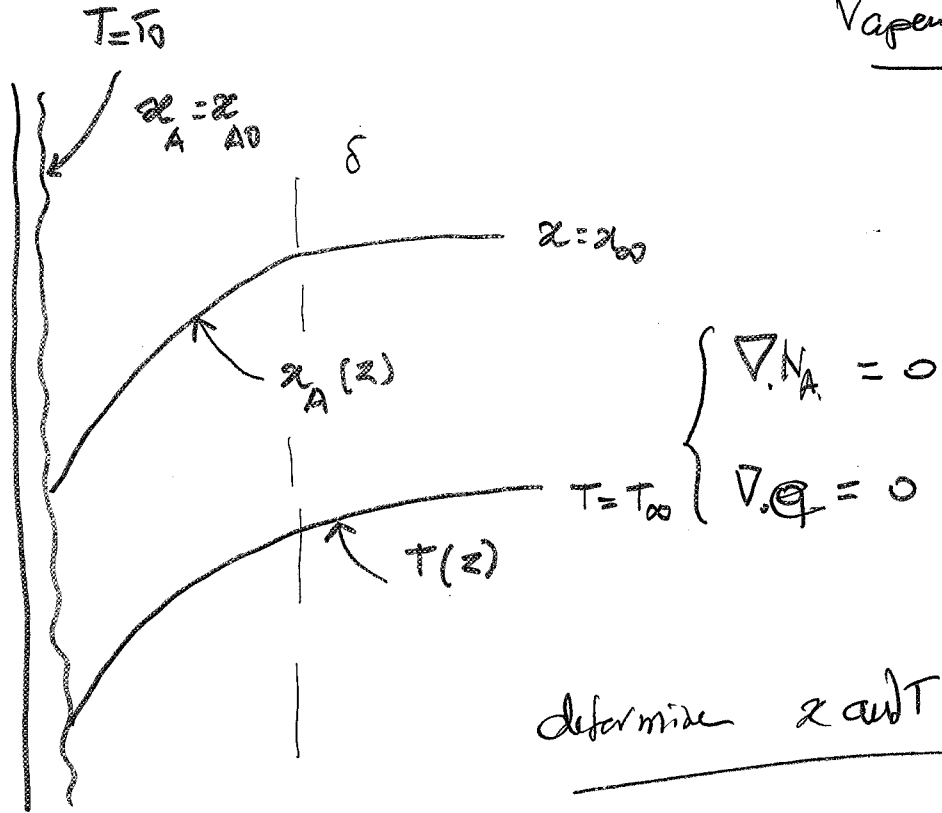
$$\omega_{vl} - \omega_{vo} > 0$$

$$B_{mi} = \frac{\omega_{vl} - \omega_{vo}}{\omega_{vo} - 1}$$



Théorie du film (Stagnant) par TDM

Vapeur + inc



$$\frac{dN_A}{dz} = 0 \quad N_A = \alpha_A N - c D_{AB} \frac{dx_A}{dz} \quad ; \quad N_B = 0.$$

$$N_A (1 - x_{A0}) = - c D_{AB} \frac{dx_A}{dz}$$

$$\int_{x_{A0}}^{x_A} \frac{dx_A}{1 - x_A} = - \frac{N_A}{c D_{AB}} \int_0^z dz$$

$$\ln \frac{1 - x_A}{1 - x_{A0}} = \frac{N_A \cdot z}{c D_{AB}}$$

$$\frac{(1-x_A)}{1-x_{A0}} = \frac{\bar{x} - x_{A0} + x_{A0} - x_A}{(1-x_{A0})} = 1 + \frac{x_{A0} - x_A}{(1-x_{A0})} = \exp \frac{N_A z}{CD_{AB}}$$

$$\frac{x_A - x_{A0}}{1-x_{A0}} = 1 - \exp \frac{N_A z}{CD_{AB}}$$

$$N_A = \frac{CD_{AB}}{\delta} \ln \left( \frac{1-x_{A0}}{1-x_A} \right)$$

$$\frac{x_A - x_{A0}}{x_A - x_{A0}} = \frac{1 - \exp(N_A z / CD_{AB})}{1 - \exp(N_A \delta / CD_{AB})}$$

profile NL  
except  $N_A \rightarrow 0$

temp.

$$\nabla \cdot e = 0$$

$$e = -k_p \nabla T + \sum_{\alpha} \bar{J}_{\alpha} \bar{H}_{\alpha} + \rho v \hat{H}$$

$$\text{or } \bar{H}_{\alpha} = \bar{h}_{\alpha} = \tilde{C}_p (T - T_0)$$

↑ averaged.

$$\rho \hat{H} v = c \tilde{H} v = v \sum_{\alpha} c_{\alpha} \bar{H}_{\alpha}$$

$$e = -k_p \nabla T + \sum_{\alpha} N_{\alpha} \bar{H}_{\alpha} \rightarrow // z$$

$$e_0 = -k \frac{dT}{dz} + (N_A \tilde{C}_{pA} + N_B \tilde{C}_{pB}) (T - T_0)$$

↑ stagnant

$$e_0 = -\frac{kT}{dz} + N_A \tilde{C}_{pA} (T - T_0) ; \frac{d}{dz} e_0 = 0$$

$$-k r^2 + r N_A c_{pA} = 0 \quad + (r - \frac{N_A c_{pA}}{k}) = 0$$

$$-k \frac{d^2 \theta}{dz^2} + N_A c_{pA} \frac{d\theta}{dz} = 0 \quad \theta = (T - T_0)$$

$$\theta = 0 \quad z = 0$$

$$\theta = (T_{\infty} - T_0) \quad z = \delta$$

$$\theta = A + B \exp \frac{N_A c_{pA} z}{k}$$

$$\begin{cases} A + B = 0 \end{cases}$$

$$B = -A$$

$$\begin{cases} A + B \exp \frac{N_A c_{pA} \delta}{k} = (T_{\infty} - T_0) \end{cases} \cdot A (1 - \exp \frac{N_A c_{pA} \delta}{k}) = (T_{\infty} - T_0)$$

$$T - T_0 = (T_{\infty} - T_0) \frac{1 - \exp(N_A c_{pA} z / k)}{1 - \exp(N_A c_{pA} \delta / k)}$$

judge w/L  
sign of  $N_A \rightarrow 0$

at the wall

$$-k \left. \frac{dT}{dz} \right|_{z=0} = \frac{(T_{\infty} - T_0) N_A c_{pA}}{1 - \exp(N_A c_{pA} \delta / k)}$$

$$\exp \epsilon \sim 1 + \epsilon + \frac{\epsilon^2}{2}$$

$$1 - \exp \epsilon \sim -\epsilon - \frac{\epsilon^2}{2}$$

$$N_A \rightarrow 0$$

$$- \frac{(T_{\infty} - T_0) \delta}{k}$$

$$- \frac{\epsilon}{-\epsilon - \frac{\epsilon^2}{2} + O(\epsilon^3)}$$

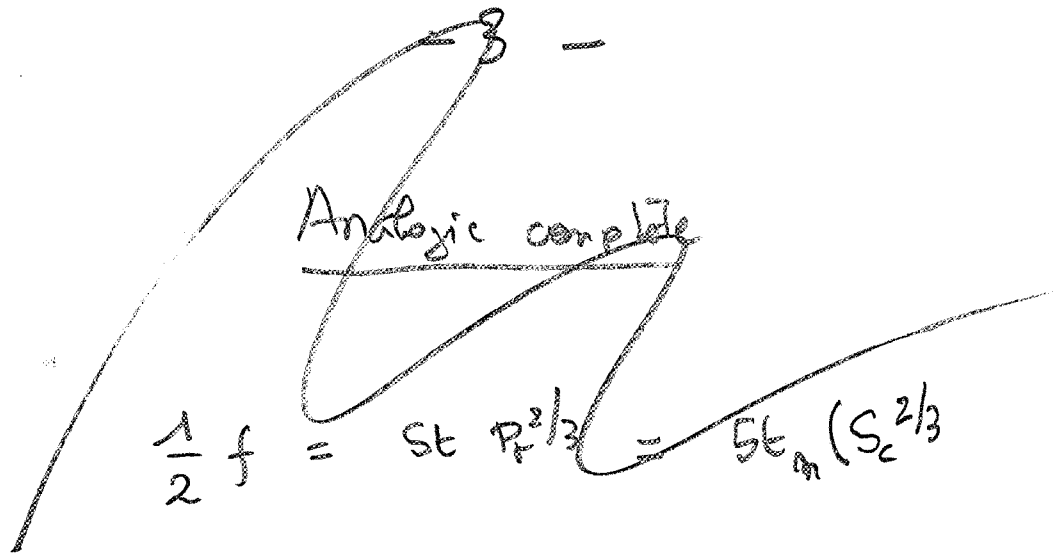
$$\frac{-k (dT/dz)_{z=\delta}}{-k (dT/dz)_{z=0}} = \frac{-(N_A c_{pA} \delta / k)}{1 - \exp N_A c_{pA} \delta / k} > 1$$

$$\frac{1}{1 + \epsilon/2 + O(\epsilon^2)}$$

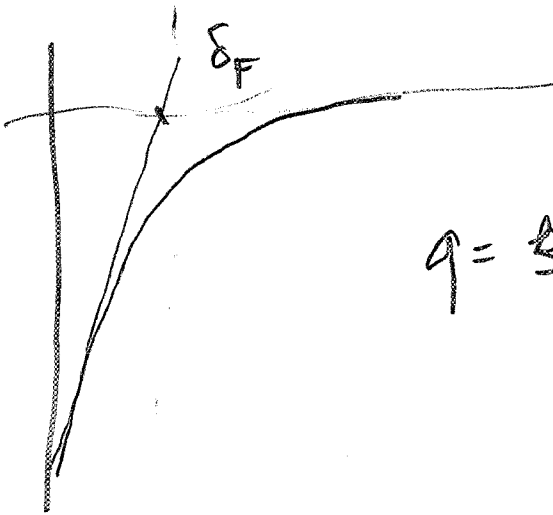
cond  $N_A < 0$

$$1 - \frac{\epsilon}{2} + O(\epsilon^2)$$

Condens. HIT - increase H Flux



$$Nu = \frac{hL}{k} = \frac{qL}{k\Delta T}$$



$$q = \frac{k\Delta T}{\delta_f}$$

$$Nu = \frac{L}{\delta_f}$$

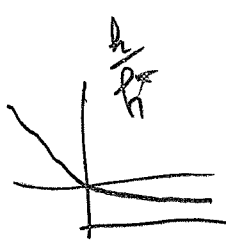
o

→  $r_h$

Condition  $\rightarrow$   $\frac{h}{k} < 0$

19.4.16  $\rightarrow$  def wcf T  
19.4.17

$$\frac{h}{h^*} = \frac{-\tau_h}{1 - \exp(\tau_h)}$$



$$\tau_h = \frac{N_A C_{PA} \delta}{k} = \frac{N_A C_{PA} (T_{\infty} - T_0)}{k (T_{\infty} - T_0)} = \frac{N_A C_{PA} (T_{\infty} - T_0)}{h^* C_{pV} (T_{\infty} - T_0)}$$

$$= \frac{N_A \phi^*}{h^* C_{pV}}$$

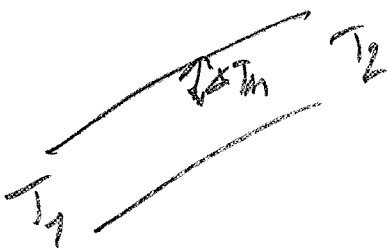
$$\frac{q_{mv}}{q_{mv}^*} = \frac{\ln(1 + B_{vm})}{B_{vm}}$$

$$B_{vm} = \frac{w_{v\infty} - w_{v0}}{w_{v0} - 1} = \frac{w_{\infty} - w_0}{w_0}$$

$\downarrow$  ircond.

Rémetat analyse pour le traitement

$$\frac{z_w}{z_w^*} = \frac{-\tau_m}{1 - \exp(\tau_m)} ; \tau_m = \frac{m \mu_{ob}}{z_w^*}$$



Vermer & Dohse

$$N C_p \Delta T = Q = q S$$

$$A G C_p \Delta T = q S \quad \Delta T_m -$$

$$\left( \frac{\Delta T}{\Delta T_m} \frac{A}{S} \right) = \frac{q}{G C_p} = \frac{R}{\alpha G} = St \quad \text{coth} \tau_m$$

def du Nbr de Sherwood.

Nusselt

flux dif.

flux dif.

$j_A$  échelle diffusion

$q$

$$\frac{\rho D_{AB} (w_{AS} - w_{AE})}{L}$$

L

$$\frac{h (T_S - T_E)}{L}$$

def du coef. de T

$$j_A = g_{MA} (w_{AS} - w_{AE})$$

$$q = h (T_S - T_E)$$

$$Sh \triangleq \frac{g_{MA} L}{\rho D_{AB}}$$

entropies  
Coef. TAM

$$Nu = \frac{h L}{k}$$

$$= \frac{j_A L}{\rho D_{AB} (w_{AS} - w_{AE})}$$

flux dif.

$$R_A = g_{MA} \frac{w_{AS} - w_{AE}}{1 - w_{AS}}$$

$$= \frac{R_A L (1 - w_{AS})}{\rho D_{AB} (w_{AS} - w_{AE})}$$

flux total.

P223

- 2

# Stagnant film model

Vorr  
Omn NACA TN 3208  
(1955) Mickley, Goss  
Spuyers Stewart.

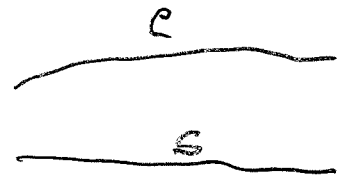
- correction for high transfer  
take BSL p. 303

- Vernier Dalbe 1984 p. 41  $\theta_d = \frac{\ln(1+R)}{R}$

- Kenhardt p. 658

$$\theta_d = \ln \frac{1+B_{mi}}{B_{mi}}$$

$$R = \frac{q_m(\omega_i - \omega_\infty)}{J_i} = \frac{\omega_\infty - \omega_i}{\omega_i} ; B_{mi} = \frac{\omega_{i,e} - \omega_{i,s}}{\omega_{i,s} - 1}$$



- theory du film <sup>BSL</sup> ex 19.4-1 p. 592

- Conv part BSL p. 346